

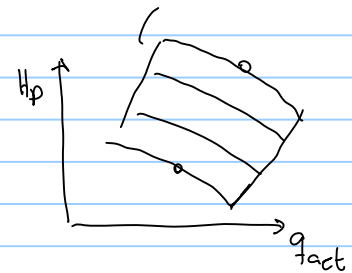
- More discussion on exercise set 3.
- Class exercise P, T with multiphase flow.

Problem 1 - Task 3

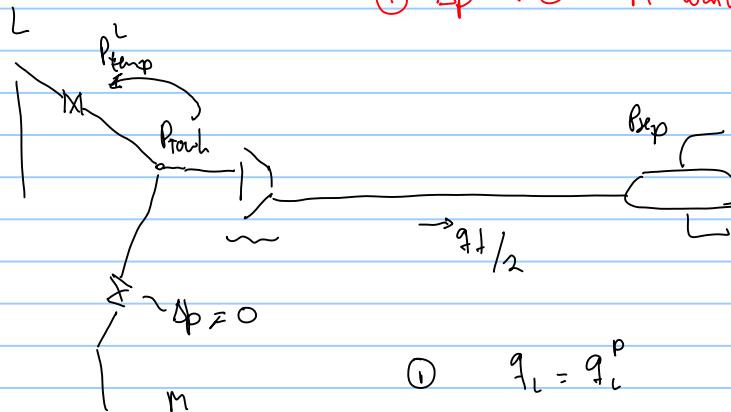
→ operational map H_p, q_{actual} ✓

→ $T_{\text{max}, \text{du}} = 120^\circ \text{C}$ ✓

→ Power $\leq 12 \text{ MW}$ ✓



① $\Delta p^m < 0$ m wants to enter in decline



$$\textcircled{1} \quad q_L = q_L^p \\ q_m = q_m^p$$

$$\textcircled{2} \quad P_r^L \rightarrow P_{wh}^L \quad \text{co-current}$$

$$P_n^m \rightarrow P_{roch}^m \quad \text{co-current} \quad (\Delta p_{\text{chore}}^m = 0)$$

$$\textcircled{3} \quad \begin{array}{l} P_{roch}^m \rightarrow P_{sep}^L \quad \text{counter-current} \\ P_{sep}^m \rightarrow P_{disch} \quad \text{counter-current} \end{array}$$

$$\textcircled{4} \quad P_{wh}^L > P_{roch}^L \quad \checkmark$$

$$P_{dis} > P_{roch}$$

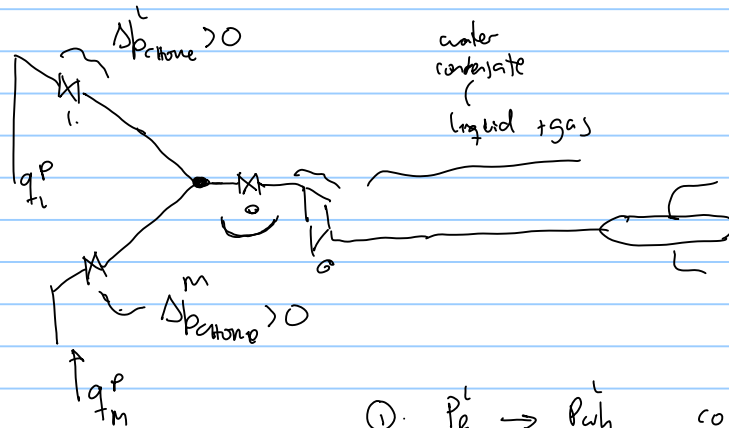
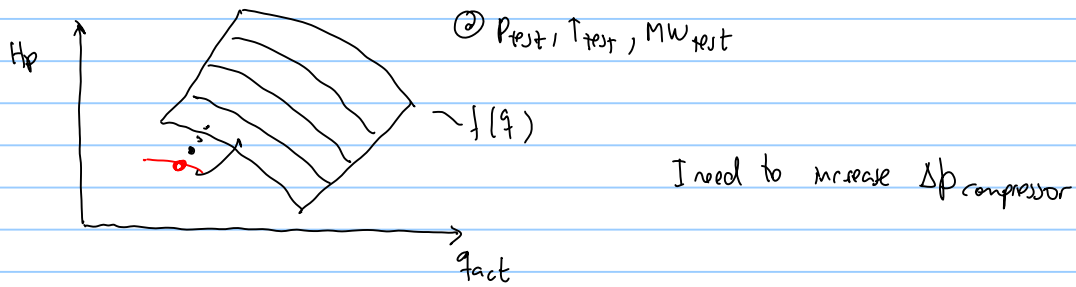
calculate

$$\textcircled{5} \quad \text{Power comp} \quad \checkmark$$

$$T_{\text{du}} = T_{\text{suc}} \cdot (r_p)^{\frac{n-1}{n}} \quad \checkmark$$

↳ [K], [°R]

$$H_p \quad q_{\text{act}} \rightarrow H_p^{\text{test}} \quad q_{\text{act}}^{\text{test}}$$



① $P_e^L \rightarrow P_{wh}^L$ co-current

② $P_e^m \rightarrow P_{wh}^m$ co-current

③ $P_{sep} \rightarrow P_{ditch}$ counter current

④ assure $\Delta p_{comp} \rightarrow P_{suc}$ *same as before*

assure $\Delta p_{partition} \rightarrow P_{twh}$

⑤ $P_{twh} \rightarrow P_{ren}^L$ counter current

$P_{twh} \rightarrow P_{ren}^m$ counter current

⑥ $\Delta p_{partition}^m > 0$

$\Delta p_{partition}^L > 0$

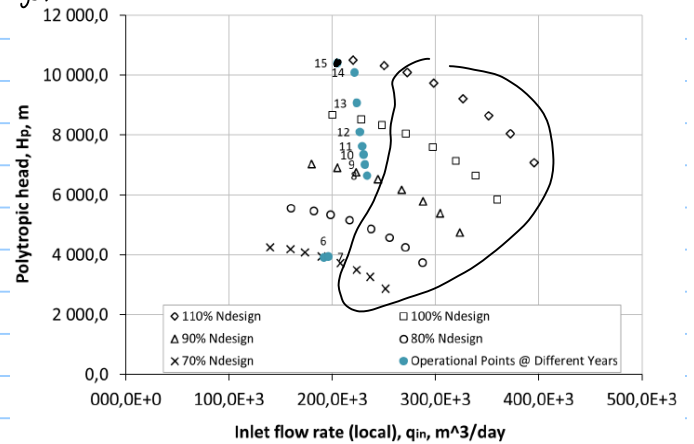
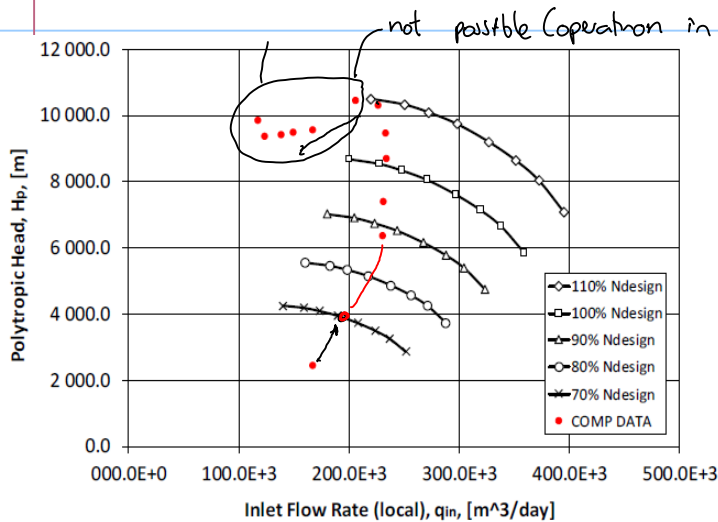
Power

T_{ditch}

$H_p^{res}, q_{act}^{test}$

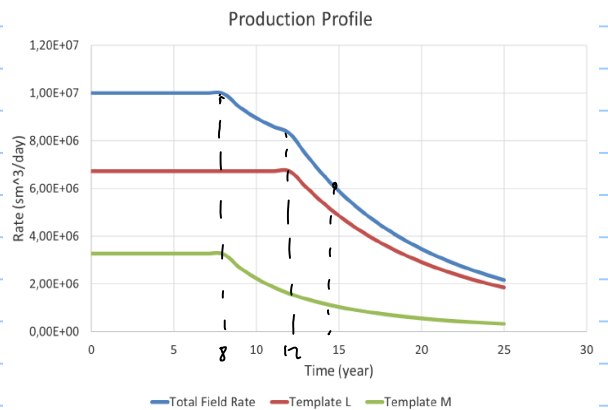
time sequence ↓

- $\Delta p_{compressor} < 0$
- both templates enter in decline

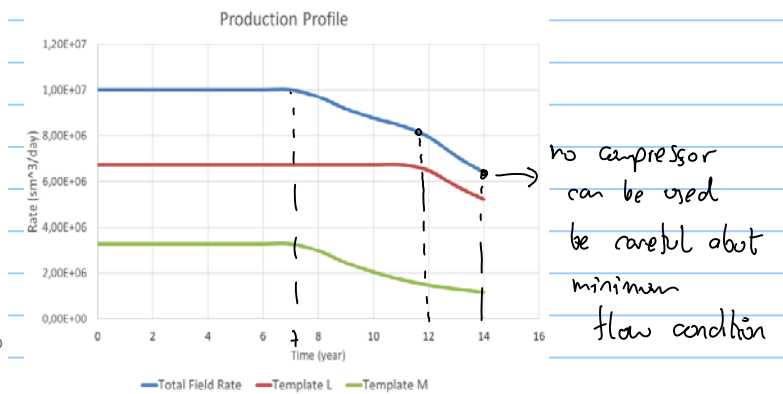


for our exercise the limit in compression power is reached (3-4) after compressor startup

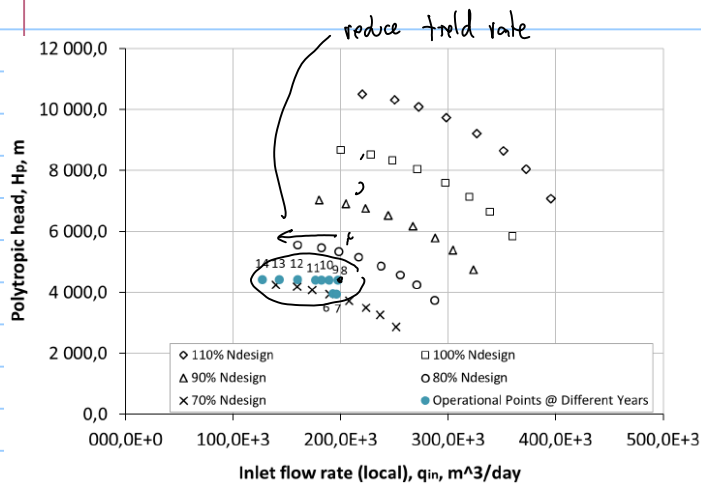
Task 2



Task 3

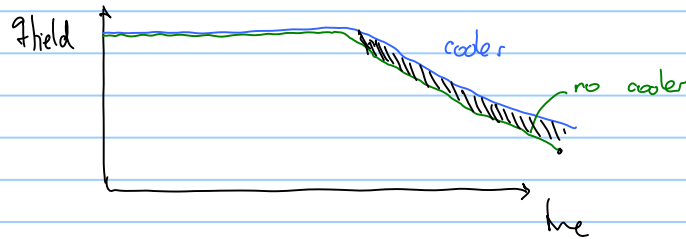


Effect of cooler if the cooler is removed $\rightarrow$ Tdu become a limitation
the q_{field} must be reduced



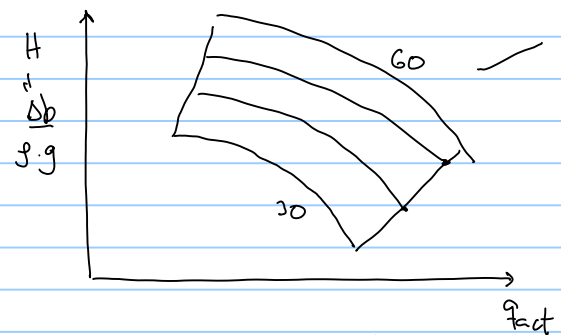
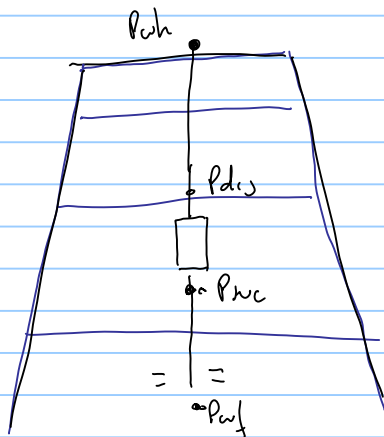
$q_{field} \downarrow \rightarrow \Delta p_{comp} \downarrow$
 $T_{disch} \downarrow \leftarrow r_p \downarrow$

no cooler gives less production



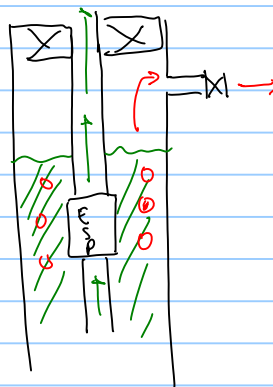
• exercise 2

$P_{aver} = 950 \text{ hp}$ ✓



$P_{wc} \geq P_b$ ✓

↳ all fluid is produced through tubing



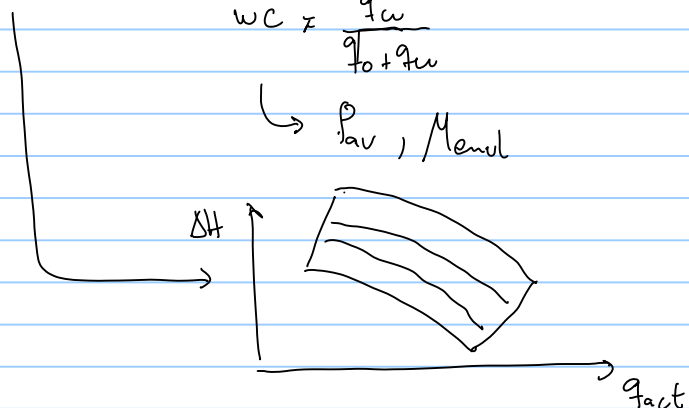
① exclude pump model
ESP maps are provided
↳ P, M

two methods

given
 q_o
 q_w

$wc \approx \frac{q_w}{q_o + q_w}$

↳ P_{av}, M_{enul}



- co-current $P_e \rightarrow P_{wf} \rightarrow P_{suc}$

- counter-current $P_{wh} \rightarrow P_{dsc}$

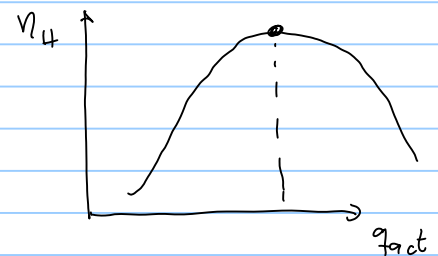
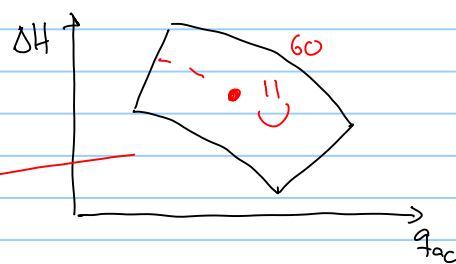
$$\Delta p_{esp} = P_{dis} - P_{suc}$$

$$\Delta H_{sp} = \frac{\Delta p_{esp}}{\rho_w \cdot g}$$

$$q_{ac} = q_e \quad \text{neglecting gas, assuming incompressible}$$

if NOT ! $q_{act} = B_o(P_{suc}, T_{suc}) \cdot q_o + B_w(P_{suc}, T_{su}) \cdot q_w$

• plot the point



check $P_{suc} \geq P_b$ (bubble point pressure at T_{suc})

$$\text{Power} = \frac{\Delta p_{esp} \cdot q_{act}^{wc}}{\eta_m \eta_H}$$

$$\eta_H = f(q_{act}, f, M, \rho)$$

estimate f graphically

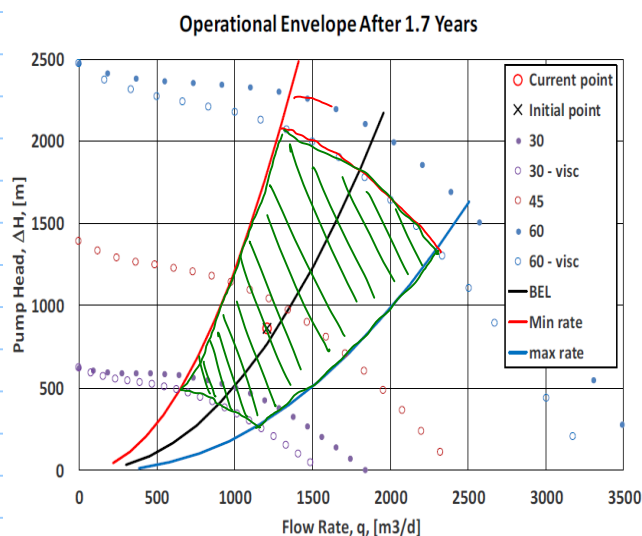
f_{ref}

q_{BEP}

best efficiency point

use UBA function $\text{pump_eff_visc}()$

use $\left\{ \begin{array}{l} \text{coefficients } \Delta H \text{ vs } q \\ \eta \text{ vs } q \end{array} \right.$ for water !



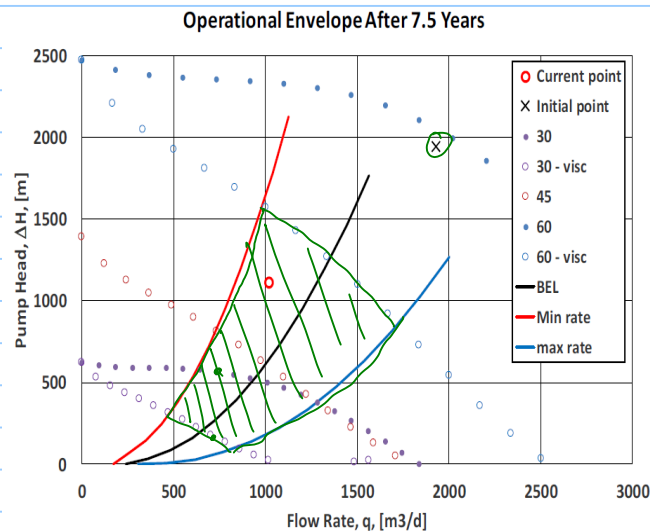
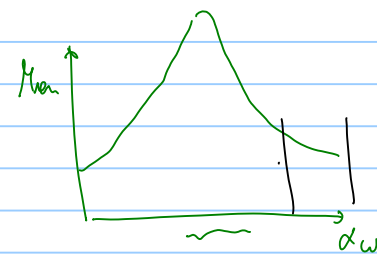
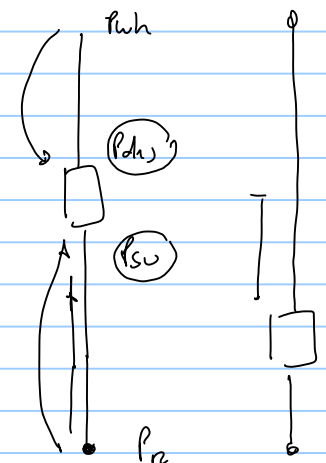
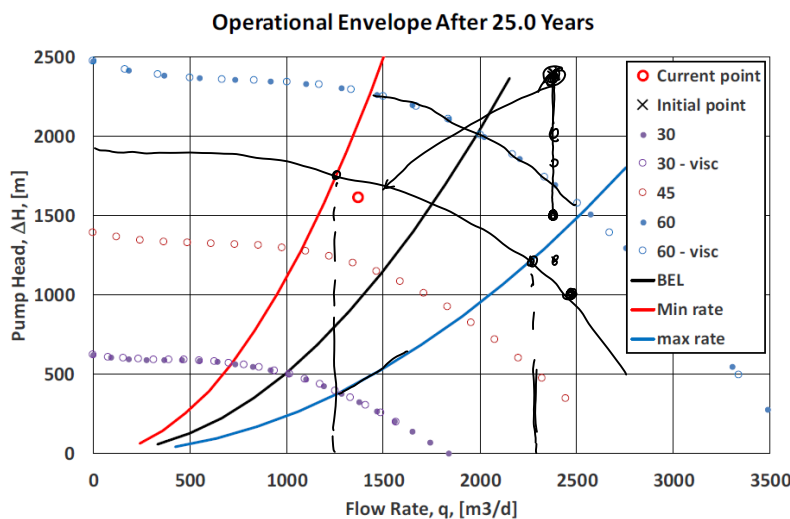


Figure 7 - The operational envelope of the ESP after 7.5 years



$$P_{suct} < P_b$$

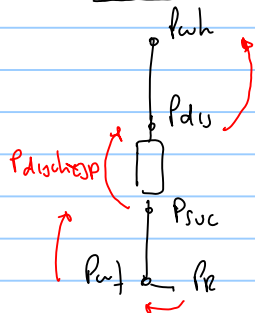


- Increase "J" → $P_{wc} \uparrow$
- $\Delta p_{pump} \downarrow$



- change ESP model
- change ESP motor (bigger)

Method 2: use the pump model (equation) (VBA function $P_{discharge_pump_visc}$)



using f as a "guess"

$$P_{wh} \neq 7 \text{ bara}$$

change f until $P_{wh} = 7 \text{ bara}$

if solver fails

numerical reasons

non feasible solutions

$\rightarrow$ as-a, for water
 $\rightarrow$ q_{BEP}
 C_q
 C_H
 starting speed
 N_{it}
 L_{TOR}

$$\Delta h_{esp} = \frac{\Delta p_{esp}}{\rho_{av} \cdot g}$$

$q_{mm} = VDA$ function

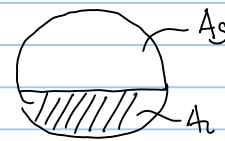
$q_{max} = VBA$ function

• class exercise, for $\frac{dp}{dx}$ calculator (multiphase aspect)

liquid superficial velocity $u_{sl} = \frac{q_L}{A}$

gas superficial velocity $u_{sg} = \frac{q_g}{A}$

A ~ cross section pipe area



<http://www.ipt.ntnu.no/~stanko/Resources.html>

-Flash-Pack, an utility based on DWSIM to estimate fluid properties, generate compositions to match measured Gas oil ratios and generate black oil tables. [Link](#)

-Black-Oil Gen, a MS Excel utility to estimate Black Oil properties and generate tables using correlations. [Link](#)

-Multiphase calculator, a MS Excel utility to estimate multiphase flow (liquid-gas) pressure drop and flow patterns. [Link](#)

Estimating well IPR with test data

The Prudhoe bay field is a large oil field on Alaska's North Slope. The field has been producing since 1977 and it is currently producing around half a million barrels per day.



The field has 1330 wells, from which 591 are naturally flowing. The wells shut-in pressure is 166 bara (corresponding to a reservoir pressure of 400 bara). The wells are arranged in production modules that provide protection and insulation to the Christmas trees from the harsh arctic environment. The various modules are connected through flowlines to the main

$$\dot{m}_T = q_o \int_o + q_g \int_g$$

 $\dot{m}_T$
 $\dot{m}_o$
 $\dot{m}_g$

with hysys

 P, T, z_i
 $\rightarrow$
 χ_o
 $\chi_g = 1 - \chi_o$
 $\frac{m_z}{m_z + m_v}$
 $\frac{m_v}{m_z + m_v}$

$$\dot{m}_o = \chi_o \cdot \dot{m}_T$$

$$\dot{m}_g = \chi_g \cdot \dot{m}_T$$