

- Methods to prolong plateau production

- Orientation for solving problem 2 exercise set 3

- single phase pressure drop
- oil water mixtures (emulsion)
- ESP constraints

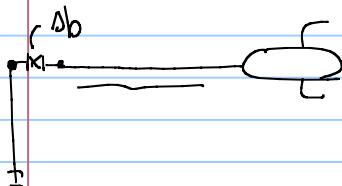
- compression?

Method to prolong plateau production:

$$1661 \cdot 10 \frac{\text{USD}}{\text{bbl}} \quad \text{Short comment on discount factor} \quad \left(\frac{1}{1+0.07} \right)^3 = 0.81 (1. \cdot 10) =$$

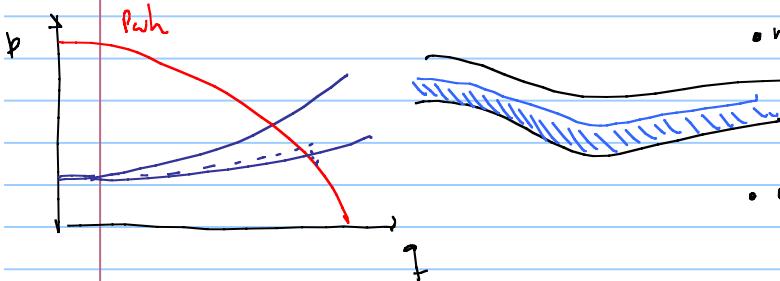
- increase number of wells (\$)

- Adding energy to system
 - boosting (compressing)
 - pumping
 - Artificial lift (ESP)
 - (\$)



- making "C" bigger fracturing, well stimulation

- making "C_L" bigger → increase tubing size
 - remove solid deposition



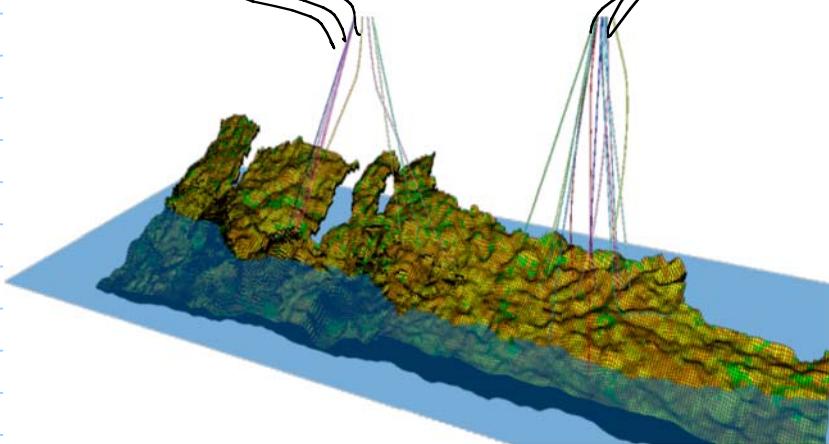
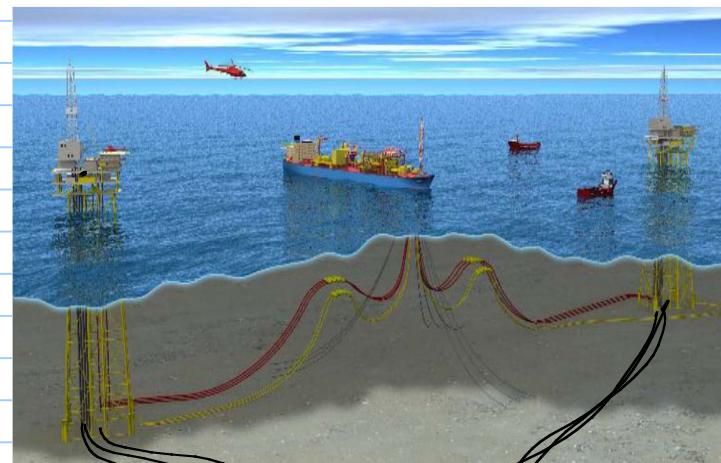
- making C_{PL} bigger → increase pipe size
 - perform pigging

- decrease P_{dep} → careful with requirement of process plant

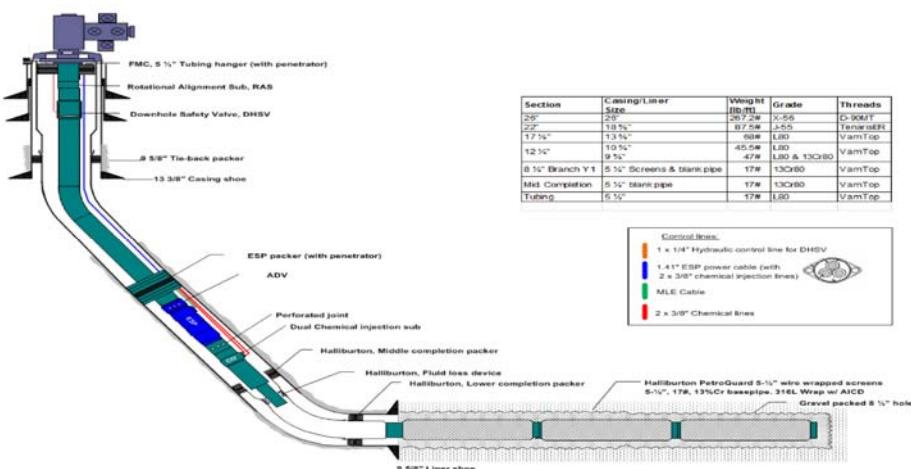
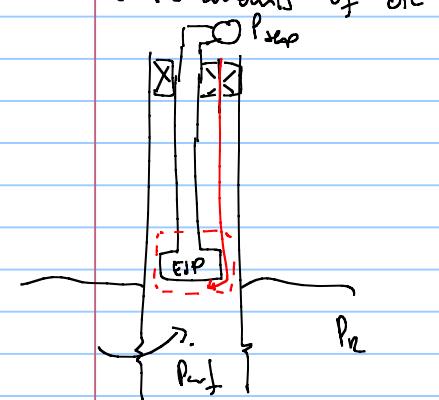
- Pressure support mechanism

- Re-distribution of production among wells
 - careful with max allowable drawdown ($P_2 - P_1$)
 - q_{max}

- Problem 2 of exercise set 3.



- the amounts of oil and water are changing with time



$$P_{injection} = P_{inlet} \text{ pump}$$

$$P_2 \rightarrow P_{wf} \quad \text{IPR equation} \quad q_{inj} = J(P_2 - P_{wf})$$

$$\textcircled{1} \rightarrow \textcircled{2}$$

$$h_1 = h_2$$

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + z_2 + h_f$$

$$P_{wf} \rightarrow P_{wuc}$$

$$q_{wf} + q_{wuc}$$

single phase and incompressible fluid

$$h_f = f \frac{L}{\phi} \cdot \frac{V^2}{2g}$$

moody friction factor

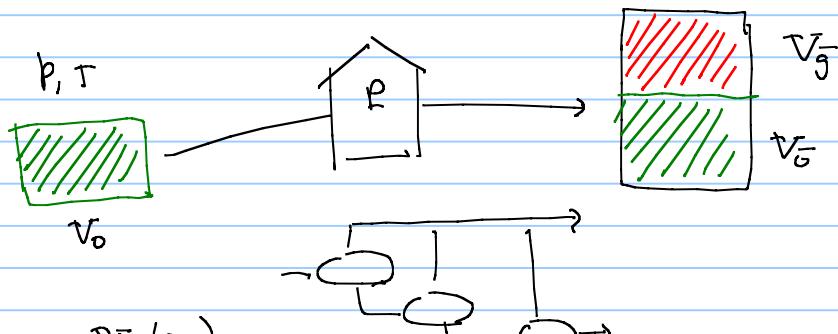
line P_1 $v_2 \approx v_1$

$$P_1 = (z_2 - z_1) \cdot \rho g + f \frac{L}{\phi} \frac{V^2}{2} \cdot \rho + P_2$$

$$V = \frac{q}{A} = \frac{q}{\pi \phi^2}$$

$$\text{eq. 1} \quad P_1 = (z_2 - z_1) \rho g + f \frac{L}{\phi} \rho \frac{8 q^2}{\pi^2 \phi^5} + P_2 \quad \begin{matrix} \text{pipe equation for} \\ \text{single phase flow} \end{matrix}$$

q is the local rate \oplus P, T is not the same as $q_o \bar{q}_w \bar{q}_i$



$$\text{oil volume factor } B_o = \frac{V_o(P, T)}{V_o(P_{sc}, T_{sc})} \rightarrow \begin{matrix} 1.8 \text{ volatile oil} \\ 1 \text{ for dead oil} \end{matrix}$$

$$\text{for this exercise } B_o = 1 \quad \text{for all } P, T$$

$$B_w = 1$$

$$q_o = \bar{q}_o$$

$$q_w = \bar{q}_w$$

$$q_i = \bar{q}_i$$

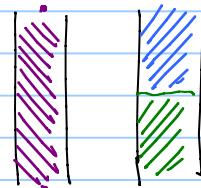
$P_{wf} \rightarrow P_{sc}$ use eq. 1 with $P_1 = P_{wf}$, $P_2 = P_{sc}$

$$q = \bar{q}_{wg}$$

$$f = f_{mix} = f_o(\alpha_o) + f_w(\alpha_w) = f_o(1-\alpha_w) + f_w \cdot \alpha_w$$

↳ volume fraction $\alpha_o = \frac{q_o}{q_o + q_w}$

$$\alpha_w = \frac{q_w}{q_o + q_w}$$



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Function to calculate average properties of an oil water mixture given the water cut
Function Avprop(WC, Po, Pv)
  "Average property Avprop
  "WC water cut (not in percentage)
  "Po property of oil
  "Pv property of water
Avprop = (WC * Pv) + ((1 - WC) * Po)
End Function

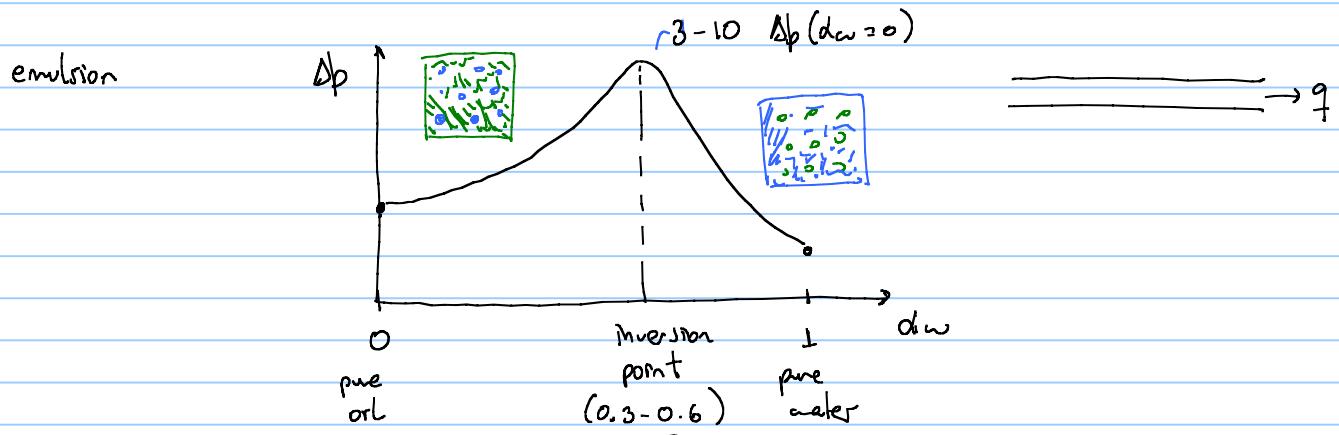
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$$\text{water at } WC = \frac{\bar{q}_w}{\bar{q}_o + \bar{q}_w}$$

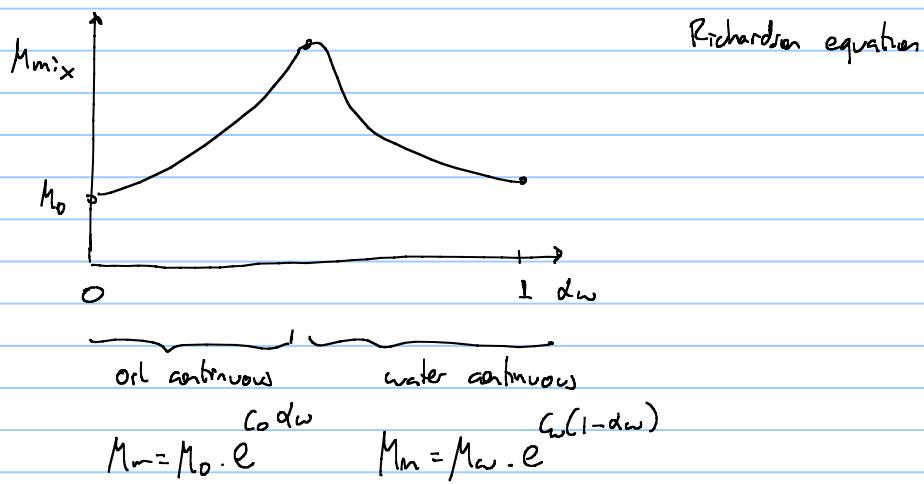
for our exercise $WC = \alpha_w$

we need viscosity for Reynolds number to estimate f

some people use $M_m = M_o (1-d_w) + M_w (\alpha_w)$ not accurate for emulsions



1 during the life of the field typically WC increases. If there is risk of emulsion this must be taken into account !



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Function Rich_emul_visc(mu_o, mu_w, alpha_w, expo, exp_w, alpha_w_cutoff)
If alpha_w > alpha_w_cutoff Then
    Rich_emul_visc = mu_w * Exp(exp_w * (1 - alpha_w))
Else
    Rich_emul_visc = mu_o * Exp(expo * (alpha_w))
End If
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End Function

$P_o \rightarrow P_{uf} \rightarrow IPR$

$P_{uf} \rightarrow P_{up} \rightarrow$ single phase Δp with P_{mix}, M_{mix}

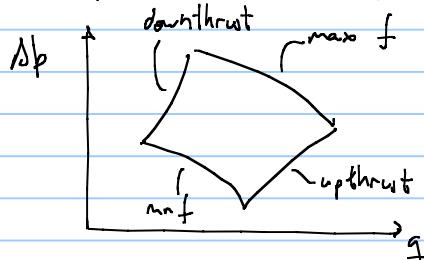
$P_{uh} \rightarrow P_{dis}$ P_{outlet} \rightarrow single phase Δp with P_{mix}, M_{mix}

method 1 : equilibrium removing EDP

how to know if the pump works? • $P_{\text{down}} > P_{\text{vac}}$

$$\bullet P_{\text{acc}} > P_b$$

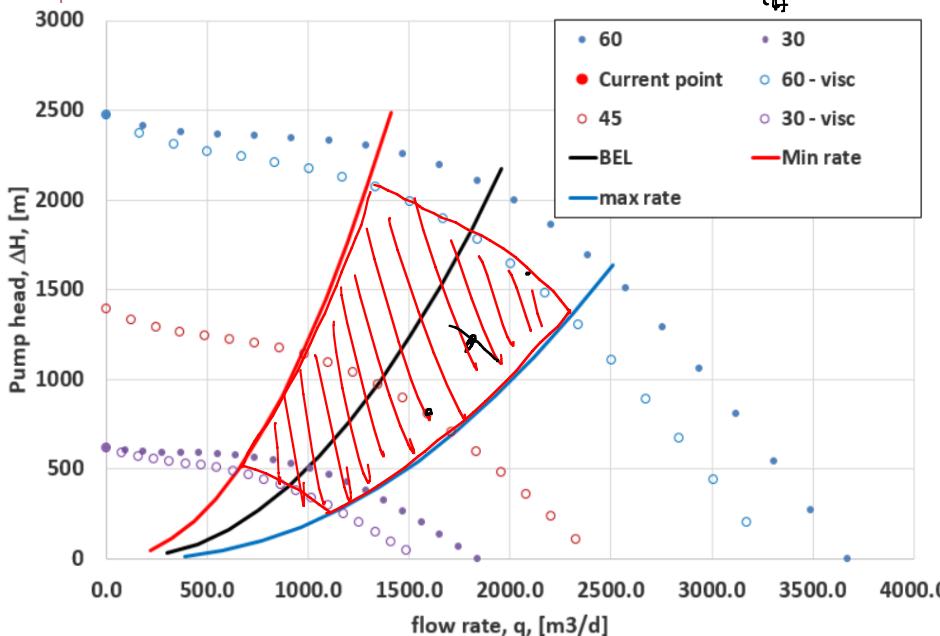
- o The operational point should fall on the operational envelope of pump



- Power required \leq Power motor
 $\leftarrow \rightarrow$ in consistent with

$$P = \frac{\Delta b \cdot g}{\eta_{H_2} \cdot \eta_m} \quad \text{assume } h_m = 1$$

$$\frac{P_c}{n_{\text{eff}}} = m^3/s$$



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$$\Delta h = \frac{\Delta b}{P}$$

the map depends on viscosity

function for hydraulic efficiency
 $q_{\text{visc}} = f$ (from interpolation)

$$N=78 \quad q_{\text{BEP}} = 198 \text{ l}$$

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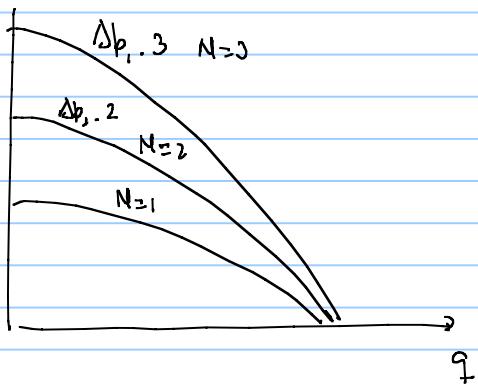
Function pumpeffic_visc(Q, f, fref, N, den, visc, Q_BEP_fref, a5, a4, a3, a2, a1, b5, b4, b3, b2, b1)
'Function to return the efficiency of the pump [in %] if the following arguments are provided:
'Q Flow through pump, [m^3/d]
'visc, fluid viscosity, [Pa s]
'Q_BEP_fref, flow rate at best efficiency point for reference frequency, [m^3/d]
'den, fluid density, [kg/m^3]
'N number of pump stages
'f pump frequency [Hz]
'fref pump reference frequency [Hz] of the original curve
'a5, a4, a3, a2, a1 fitting coefficients of the H vs Q curve to a fourth order polynomial of one ESP stage with H in m and Q in m^3/d
'b5, b4, b3, b2, b1 fitting coefficients of the effic vs Q curve to a fourth order polynomial of one ESP stage with effic in % and Q in m^3/d

Q_BEP = Q_BEP_fref * f / fref
H_BEP = pumphead(Q_BEP, f, N, fref, a5, a4, a3, a2, a1)
B = Bparameter(den, visc, H_BEP, Q_BEP, f)
If B <= 1 Then
    pumpeffic_visc = ((a5 * ((Q * fref / f) ^ 4)) + (a4 * ((Q * fref / f) ^ 3)) + (a3 * ((Q * fref / f) ^ 2)) + (a2 * (Q * fref / f)) + a1)
Elseif B > 1 And B < 40 Then
    Ceffic = B ^ (-0.0547 * (B ^ 0.69))
    CQ = 2.71 ^ (-0.165 * (Log10(B) ^ 3.15))
    QW = Q / CQ
    pumpeffic_visc = Ceffic * ((b5 * ((QW * fref / f) ^ 4)) + (b4 * ((QW * fref / f) ^ 3)) + (b3 * ((QW * fref / f) ^ 2)) + (b2 * (QW * fref / f)))
Elseif B >= 40 Then
    pumpeffic_visc = "ERROR, too viscous flow"
End If

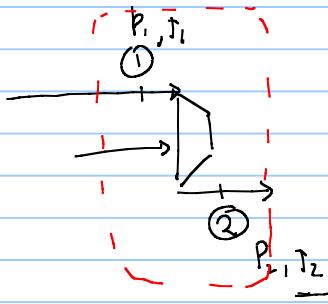
End Function
End Function

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N_b



Compression (gas)

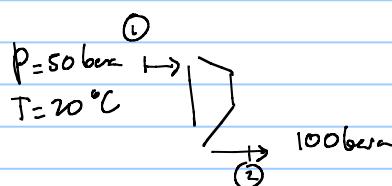


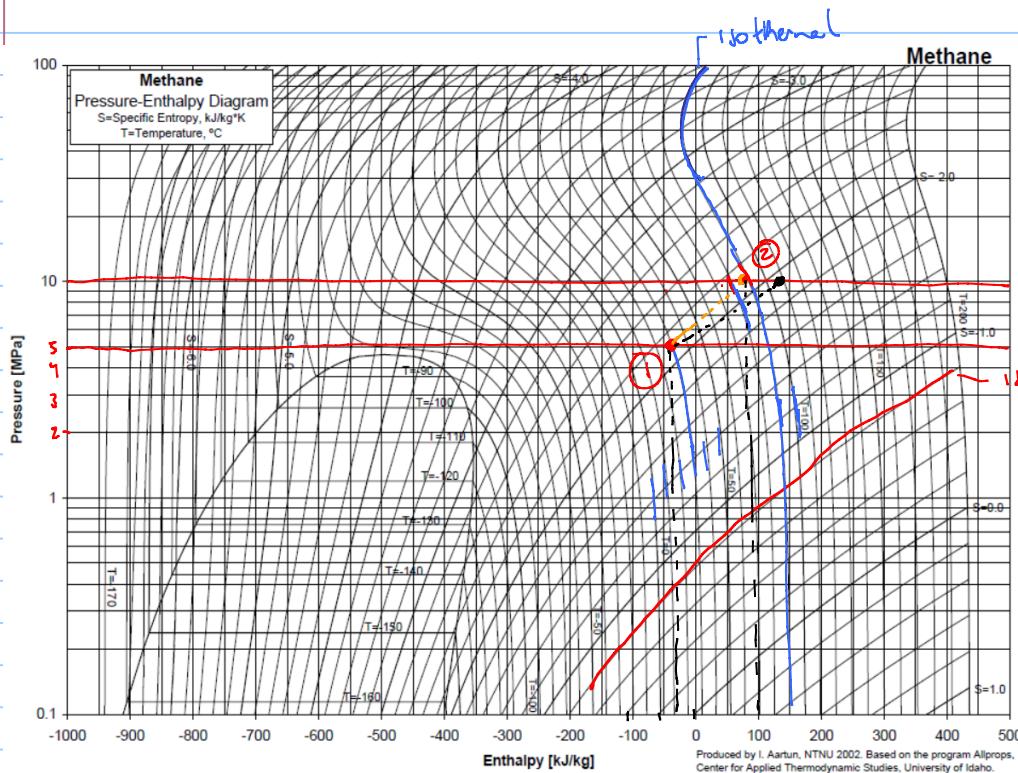
$$\dot{\omega} = \dot{m}(e_1 - e_2)$$

~~$$\dot{\omega} = \dot{m} \left(\frac{v_1}{2g} + z_1 + h_1 - \frac{v_2}{2g} - z_2 - h_2 \right)$$~~

$$\dot{\omega} = \dot{m} (h_1 - h_2)$$

for Gullfaks exercise





50 bar \rightarrow 100 bar
5 MPa

$P_{d_1} = 100 \text{ bar} = 10 \text{ MPa}$

isentropy lines

$T_2 = 85^\circ\text{C}$

$$\dot{w}_s = \dot{m}(h_2 - h_1)$$

$$\dot{w}_s = 78 \frac{\text{kg}}{\text{s}} \cdot \left(100 \frac{\text{kJ}}{\text{kg}} + 25 \frac{\text{kJ}}{\text{kg}} \right) = -983 \frac{\text{kJ}}{\text{s}} = 9.8 \text{ MW}$$

$$\dot{f}_2 = 10 \text{ g} \cdot \frac{1}{1000} \text{ kg/g}$$

$$\dot{f}_2 = 0.68 \frac{\text{kg}}{\text{s}}$$

$$\dot{m}_2 = 10 \frac{\text{kg}}{\text{s}} \cdot 0.68 \frac{1 \text{ kg}}{3600 \text{ s}} = 78 \frac{\text{kg}}{\text{s}}$$