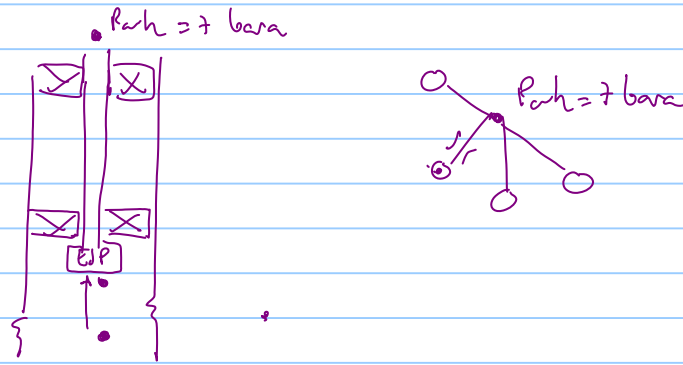


http://folk.ntnu.no/stanko/Courses/TPG4230/2017/Class_files/20170404/

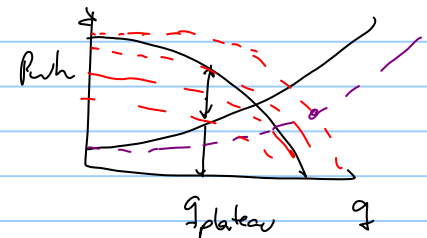


30 [bara]		Required pressure at pump intake								36								Max power [hp]		950	
P_R	WC	Average density	Effective viscosity	Q_{tot}	P_{wf}	Q_o	Q_w	P_{suc}	f	P_{disc}	ΔP_{ESP}	Δh_{ESP}	$P_{wh calc}$	$P_{wh given}$	Hydraulic Effic	Pump power	q_{min}	q_{max}			
bara	[-]	kg/m ³	Pa s	Sm ³ /d	bara	Sm ³ /d	Sm ³ /d	bara	[Hz]	bara	bara	m	bara	bara	[-]	[hp]	[m ³ /d]	[m ³ /d]			
231	0.10	910	0.138	1700	110	1530	170	75.7	53.0	188	113	1264	7.0	7	0.452	658.6	1121	1994			
215	0.10	910	0.138	1700	94	1530	170	59.6	55.3	188	129	1445	7.0	7	0.455	748.5	1169	2079			
200	0.10	910	0.138	1700	79	1530	170	44.6	57.4	188	144	1613	7.0	7	0.456	833.4	1213	2156			
175	0.10	910	0.138	1470	70	1323	147	36.0	56.5	187	151	1693	7.0	7	0.438	786.6	1194	2124			
150	0.10	910	0.138	1120	70	1008	112	36.0	54.1	185	149	1673	7.0	7	0.401	647.4	1144	2033			
125	0.10	910	0.138	771	70	694	77	36.0	52.5	184	148	1656	7.0	7	0.344	513.3	1110	1973			

Singh, B2 Tanzania what can we do to prolong plateau?

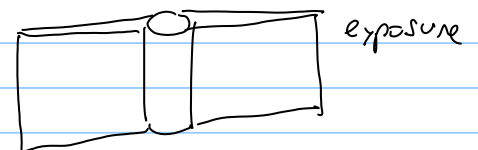
- increase available pressure $P_{wh avail}$
- decrease required pressure $P_{wh req}$

methods to increase available pressure:



• $q = \frac{C_R}{2\pi} (P_R^2 - P_{wf}^2)^n$ Increase reservoir deliverability: acidizing
injecting chemical to improve permeability of near wellbore region

• Fracturing. increase wellbore formation exposure



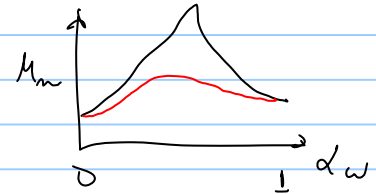
• multi-laterals
• multi-layer (activate in another segment)

• Inflow control devices.
reduce the flow of other phases that affect the flow of the main phase

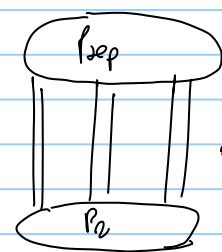
- pressure support in reservoir: water injection, gas injection \$

- tubing:
 - increase tubing ID
 - install artificial lift
 - ESP
 - PCP
 - jet pump

- reduce μ of fluid $\rightarrow$ demulsifiers
 - ↘ solvent
 - ↘ lighter API oil



- increase the number of wells

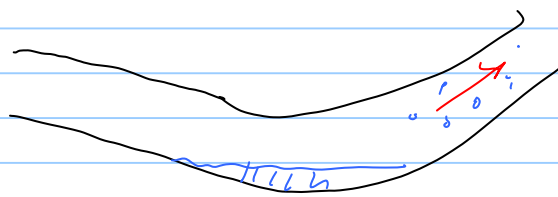


Δp in each conduit is less



reducing required pressure

- changing size (ID) of transportation flowlines and pipelines $\uparrow ID \uparrow \$$



WARNING! Be careful with liquid accumulation in gas/liquid flow.

- a parallel pipeline !
- reduce $\mu_m \rightarrow$ inject demulsifier solvent.

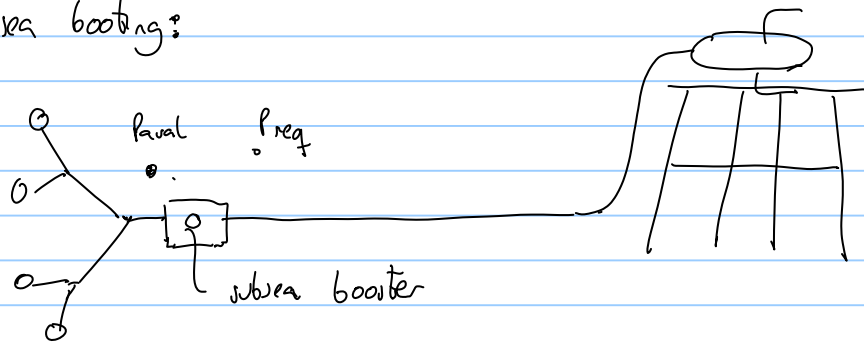
- reduce separator pressure



~ big chance in topside / processing facilities

- subsea processing: separation of oil, gas, water to avoid unnecessary transportation losses. from seabed to topside

• Subsea boosting:



2 types of booster : • single phase liquid, gas $\rightarrow$ requires separation before
• multiphase $\rightarrow$ oil, gas, water

two types of boosting principles

Positive displacement

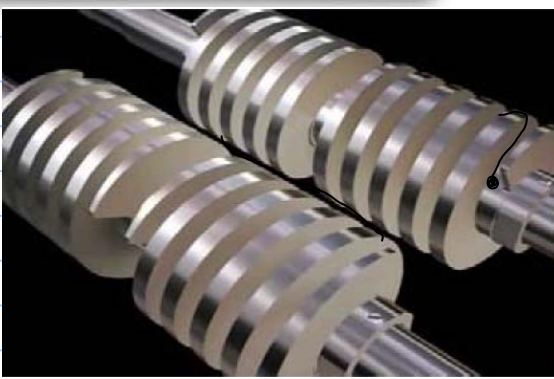
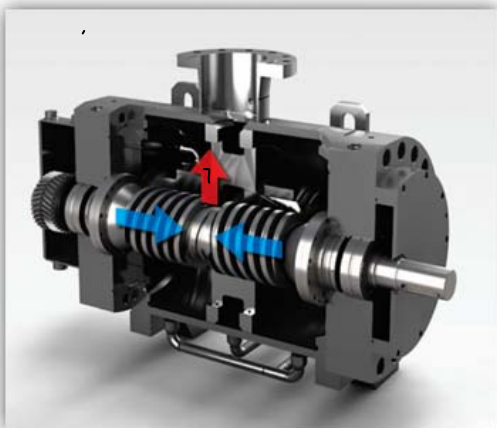
- gives higher ΔP
- don't tolerate solids
- lower rates

Rotodynamic

- gives medium $\rightarrow$ low ΔP
- have some tolerance to solids.
- medium and high rates.

multi-phase boosters

twin screw pump



HAP helico-axial pump.

Framo. OneSubsea



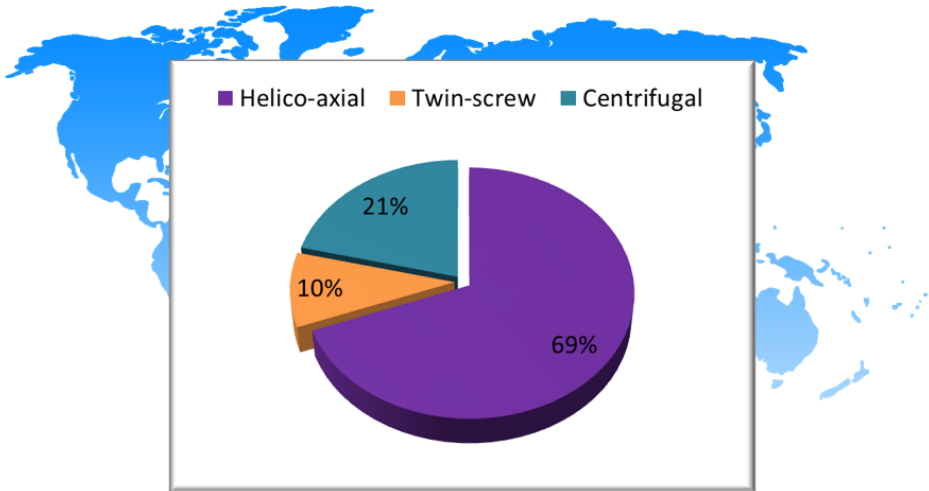
(a) Impeller



(b) Diffuser

Subsea multiphase boosting around the world

52 subsea pumps in 23 locations in the World



From <http://www.offshore-mag.com/>

Wet gas compressor $\rightarrow$ lots of gas with some liquid.
gas-liquid mixture is possible to model it as a single fluid with average properties.
counter-rotating wet gas compressor.

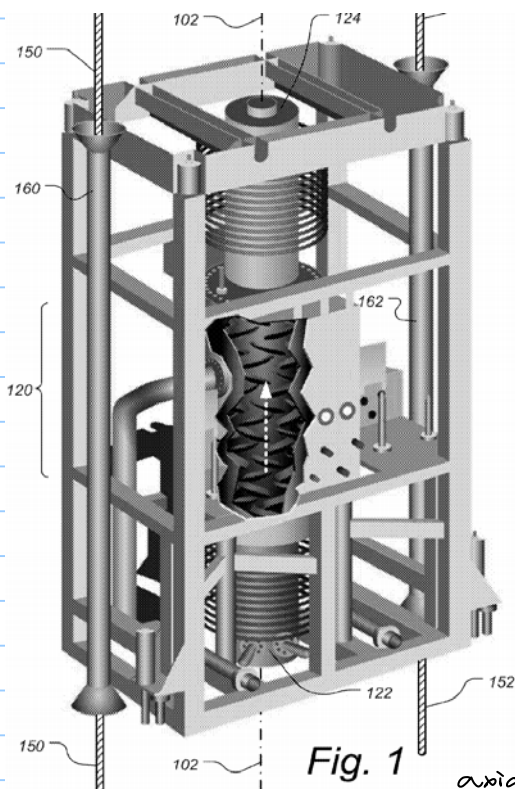
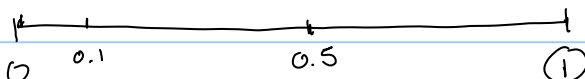


Fig. 1

axial impellers { HAP
WGC

mixed flow



radial
centrifugal
impellers

$\rightarrow$ high ΔP

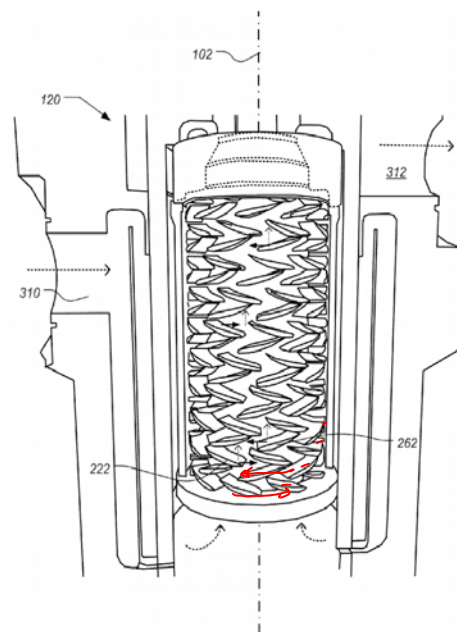


Fig. 3A

gives a lower ΔP

*usually requires
more stages
than
centrifugal
for same Δp*

GVF gas volume fraction

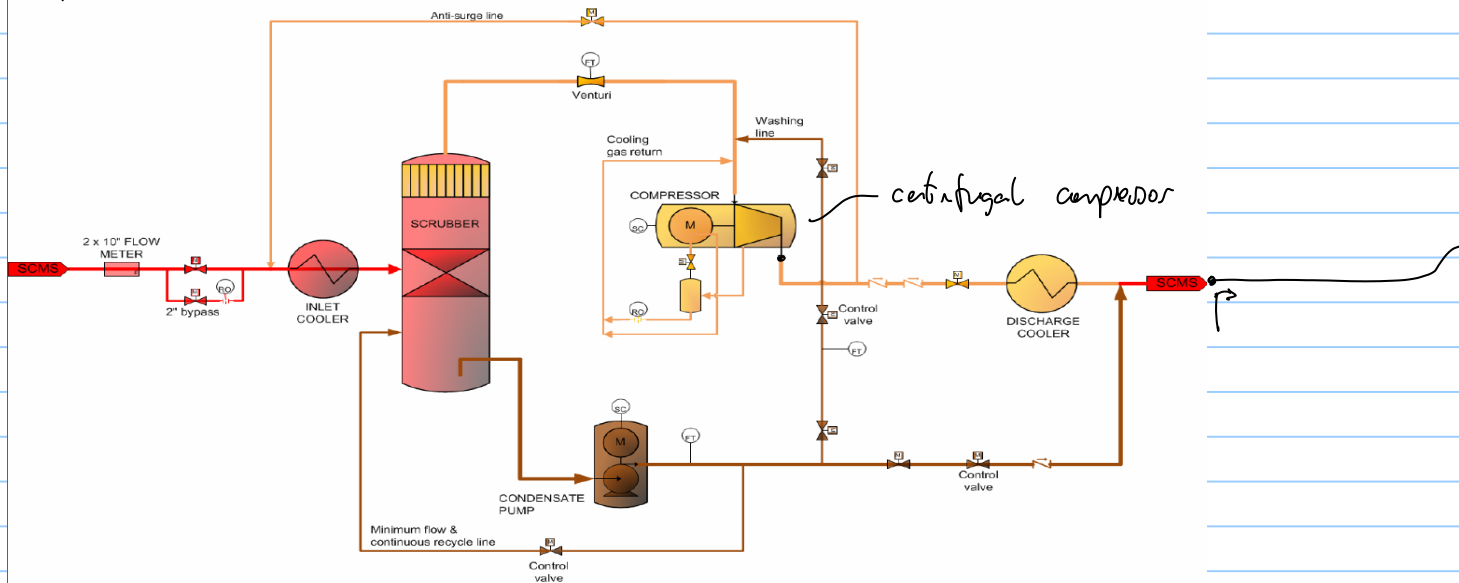
g_g

$g_o + g_w + g_g$

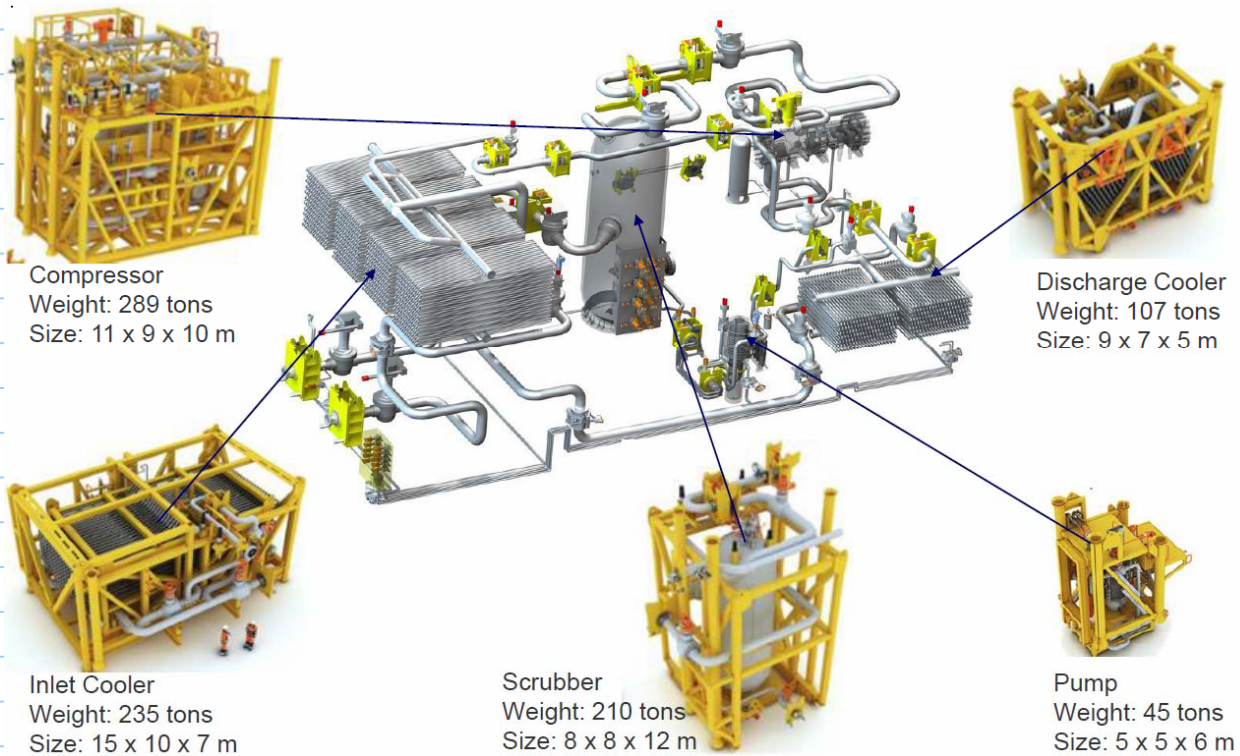
@ local conditions
batter inlet

Single phase subsea boosters → separation before! Aasgard subsea compression dry compressor + separation station

Process Flow Diagram

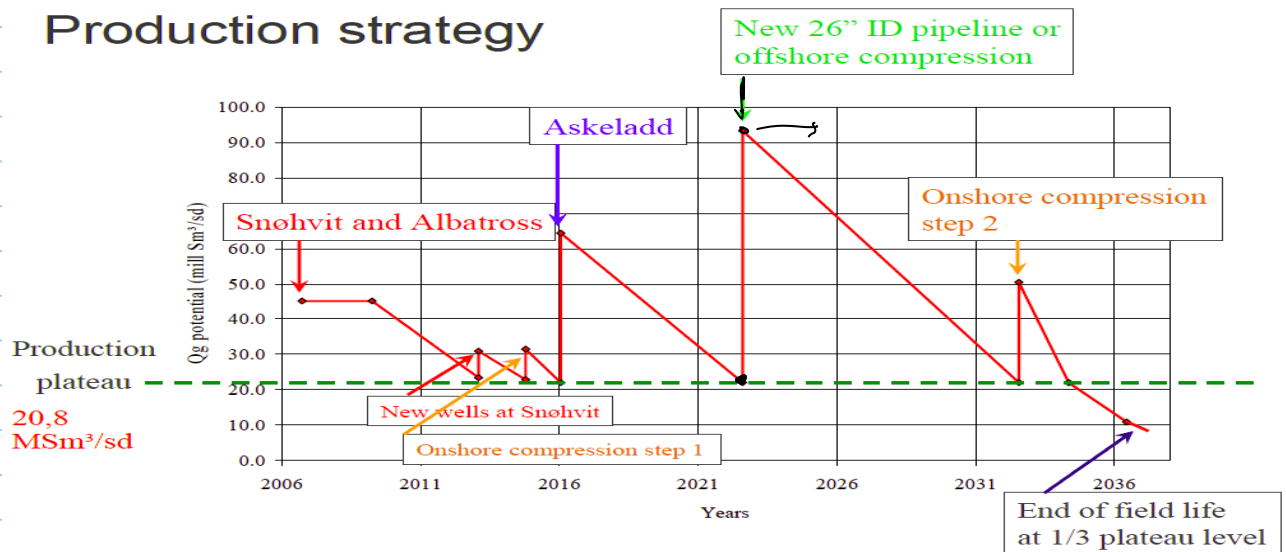


Process Modules- Sizes and Dry Weights

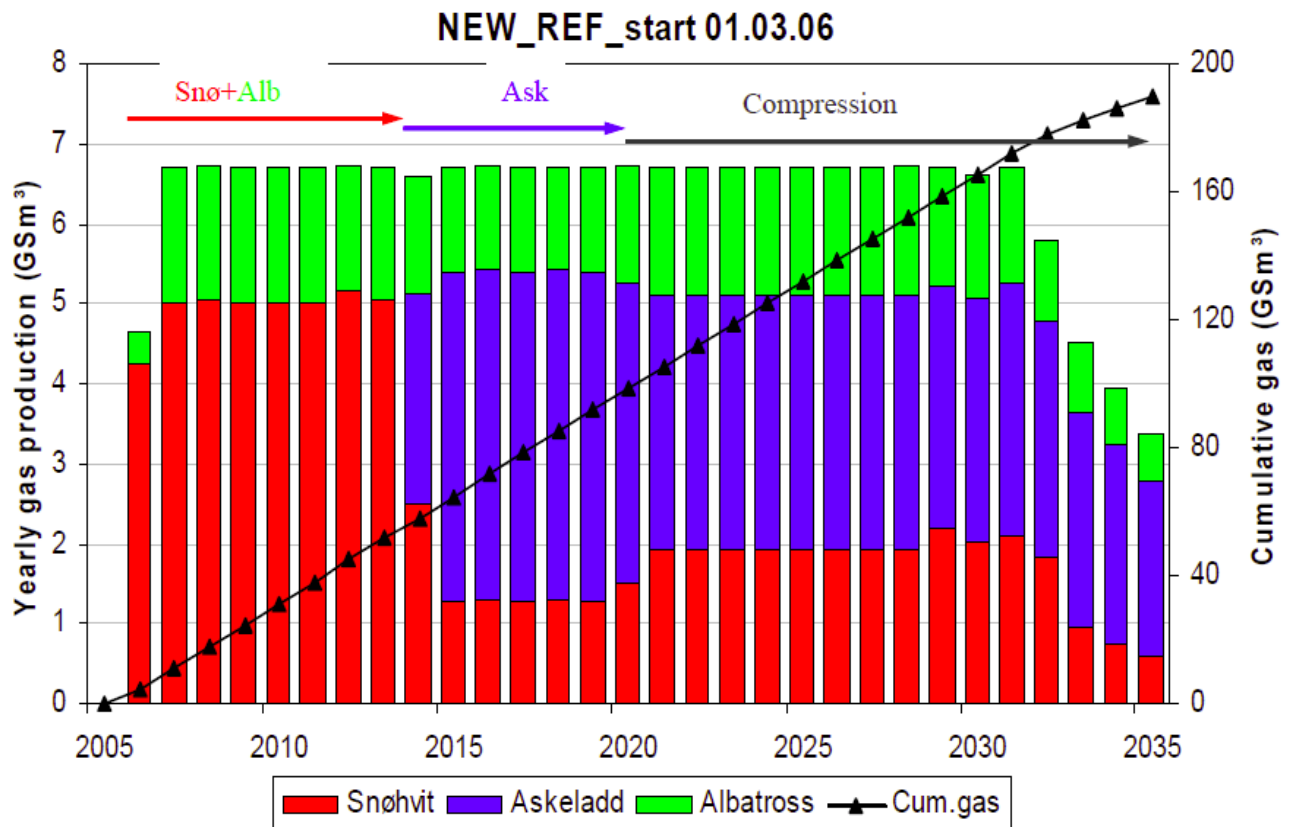


In particular, the Snøhvit field:

Production strategy



Production profile (20,8mill. Sm³/sd – 6,7 GSm³/år)

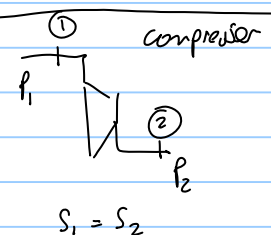


Refreshment in compressors

if I compress ideally:

$$\tilde{T}_{2s} = \left(\frac{p_2}{p_1} \right)^{\frac{\kappa-1}{\kappa}} = r_p^{\left(\frac{\kappa-1}{\kappa} \right)}$$

$$\kappa = \frac{C_p(T)}{C_v(T)} = \kappa f(T)$$



in reality the process is not isentropic!

the compression process is represented using a polytropic expansion

$$\frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}}$$

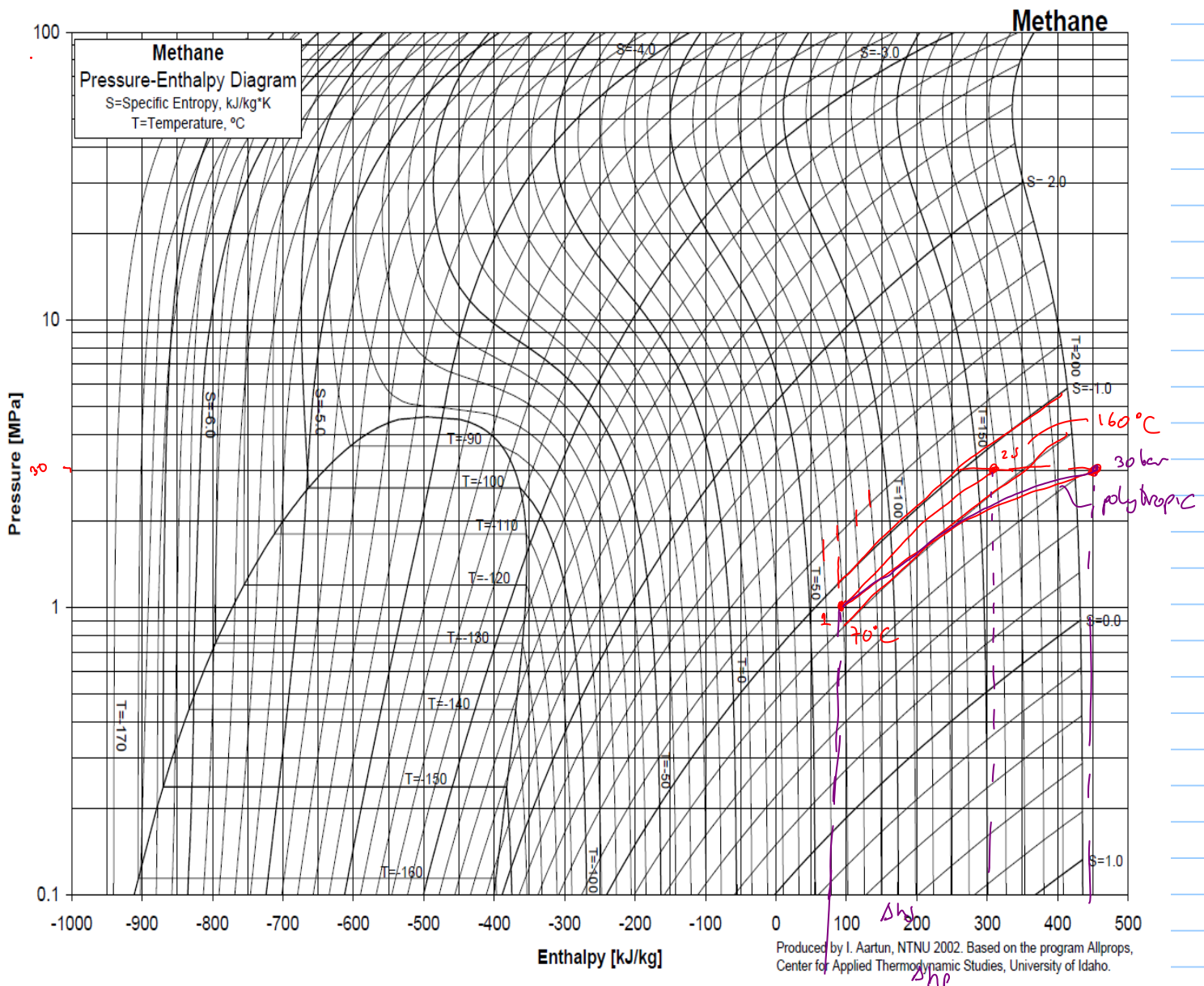
$$P V^n = \text{const}$$

polytropic exponent

the real compression power required is given by:

$$2 \phi \underbrace{(T_2 - T_1)}_{\text{for liquids}} = h_2 - h_1$$

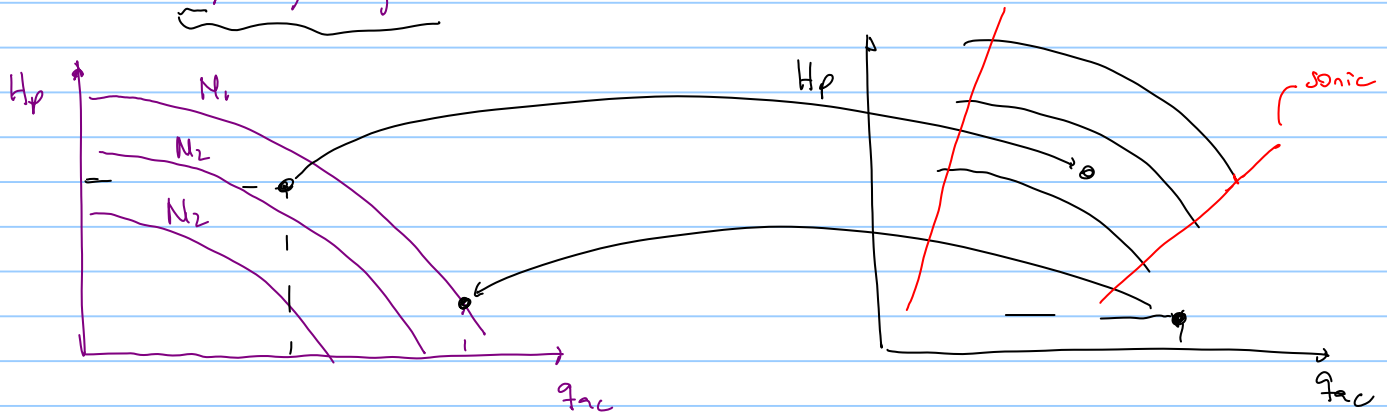
$$P_{\text{comp}} = \frac{\dot{m}' \Delta h_p}{\underbrace{\eta_p \eta_m}_{\left(\frac{n-1}{n}\right) = \left(\frac{\kappa-1}{\kappa} \eta_p\right)}} = \frac{\dot{m}}{\eta_p \eta_m} \cdot \underbrace{z_{\text{av}} \cdot \bar{R} \cdot \frac{n}{n-1} \left(\left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}} - 1\right) T_1}_{\substack{\Delta h_p \rightarrow \bar{R} = \frac{R_u}{M_{\text{Wgas}} \\ \frac{z_1 + z_2}{2}}}}$$



$$H_p = \frac{\Delta h_p}{g} \approx \text{polytropic head}$$

$$\text{for } P_1^*, T_1^*, \dot{m}_w^*, k^*$$

test the compressor at $P_1 = 1.01325 \text{ bar}$
 $T_1 = 15.56^\circ\text{C}$
 $\dot{m}_w = 20.97$
 $k = 1.3$?



to calculate compressor performance for other conditions different than test
 apply the following correction:

$$q_{act} = q_{test} \sqrt{\frac{k_{ac}}{k_{tes}}} \cdot \sqrt{\frac{\dot{m}_{w,tes}}{\dot{m}_{w,ac}}} \cdot \sqrt{\frac{T_{ac}}{T_{test}}} \quad q_{test}$$

$$H_p^{corrected} = \frac{\dot{m}_{w,tes}}{\dot{m}_w} \cdot \frac{T_1}{T_{1,tes}} \cdot \frac{k}{k_{tes}} \cdot H_p^{test}$$

Compressors constraints:

- discharge $T \leq 150^\circ\text{C}$
 - integrity pipeline
 - avoid vaporization glycol hydrate inhibitor
 - avoid damaging compressor seals

another assumption

$$\eta_p = \text{const} \approx 0.7$$

in reality

$$\eta_p = f(q_{act})$$

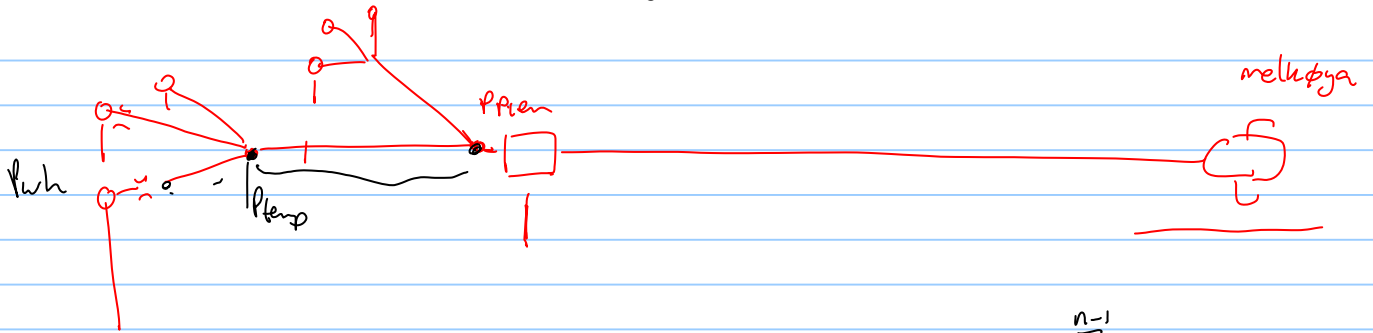
$$\text{Power} \leq P_{max} (11 \text{ MW})$$

- $P_{suc} \geq 10 \text{ bara}$
 - to activate seals
- operational speed 9350 - 6800 rpm

$$q_{min} \leq q \leq q_{max}$$

surge. $\left\{ \begin{array}{l} \text{flow} \\ \text{back and forth} \\ \text{in machine} \\ \text{(unstable)} \end{array} \right.$

sonic velocity in machine passages $M = 1$



estimate T_{outlet} $T_2 = T_p \frac{n-1}{n}$

$$\text{estimate } \delta_{hp} = \bar{z}_{av} \cdot \bar{R} \cdot \frac{n}{n-1} \left(r_p^{\frac{n-1}{n}} - 1 \right) T_1$$

calculate $H_p = \frac{\Delta h_p}{g}$

calculate $q_{act} = \underbrace{B_g(P_T)}_{\text{at } P_T} \cdot f_{sc}$

$$B_g = z \frac{T}{T_{sc}} \frac{P_{sc}}{P}$$

$H_p, f_{act} \rightarrow$ convert to test conditions
plot in compressor map.