

Artificial lift methods:

- Gas lift
- Electric submersible pump (ESP)
- PCP progressive cavity pump $\rightarrow M \uparrow$
- Rod pump $\downarrow q$

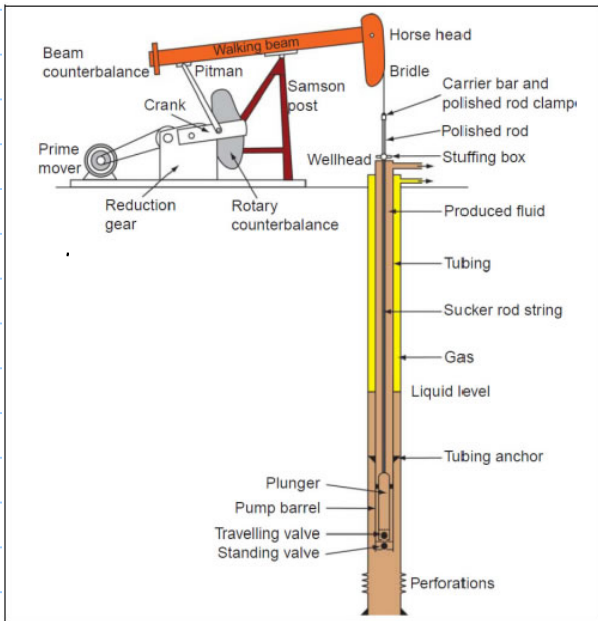
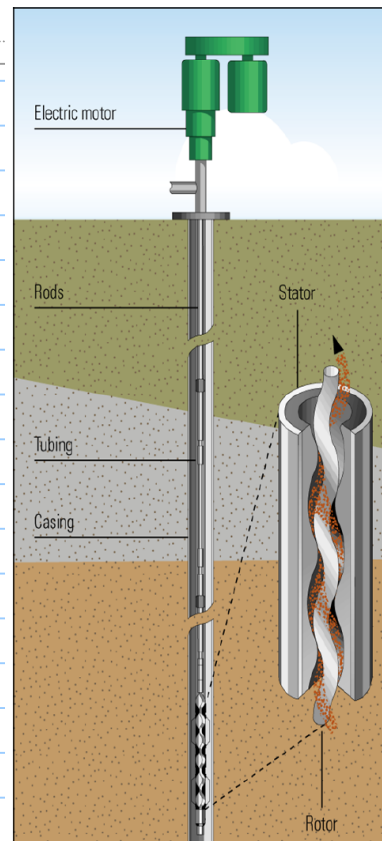
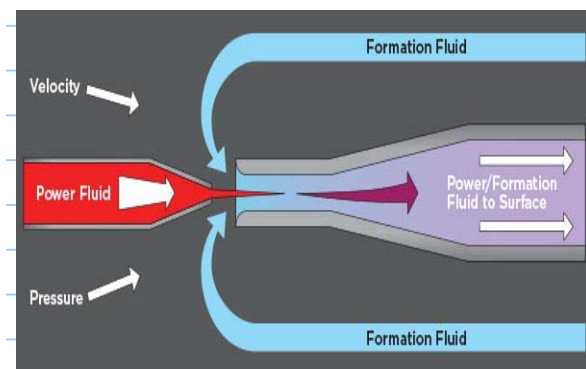
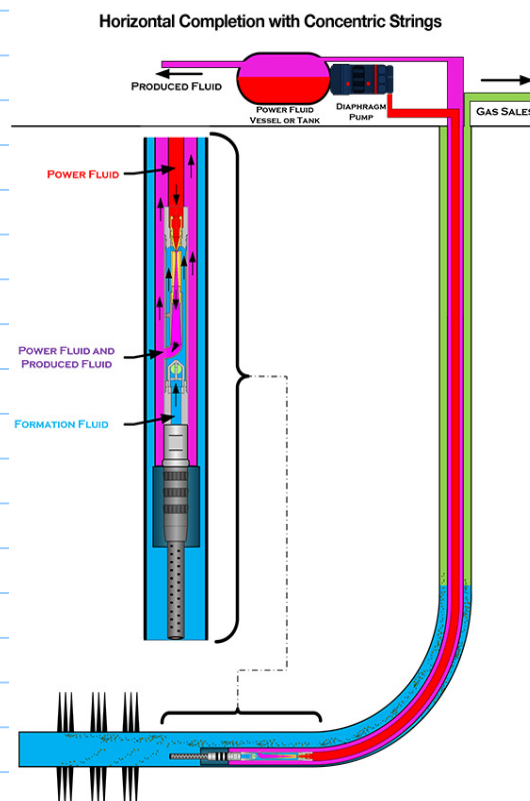


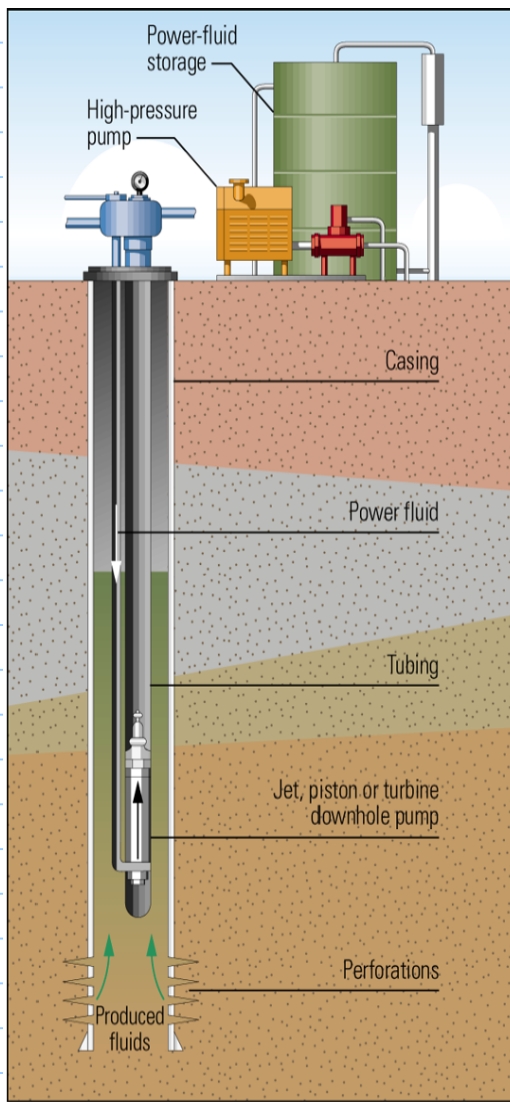
Figure 2. Typical configuration of a sucker rod pump (Bellarby, 2009).



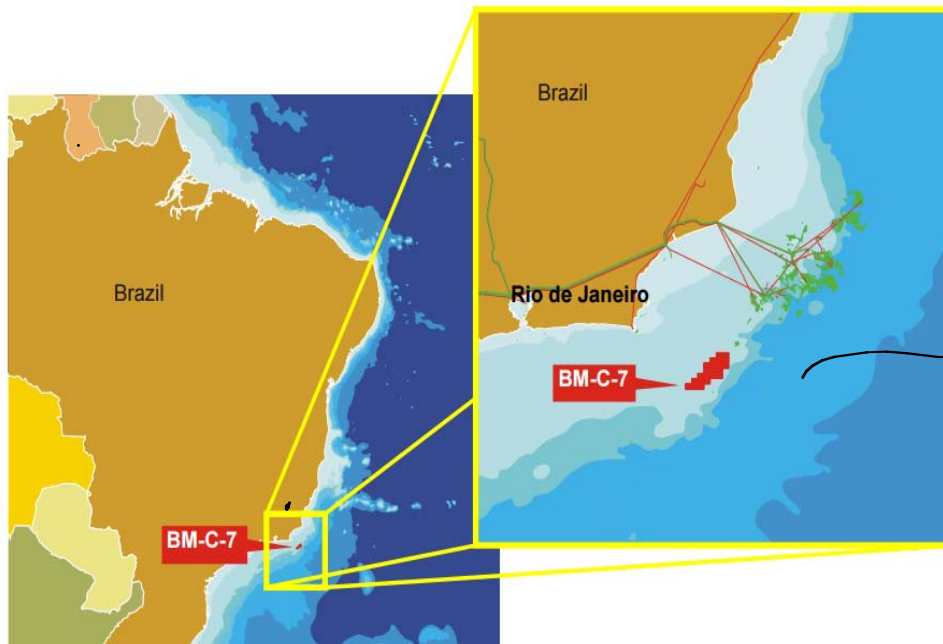
$\leftarrow$ positive displacement principle

• Jet pump





Peregrino field



100 m

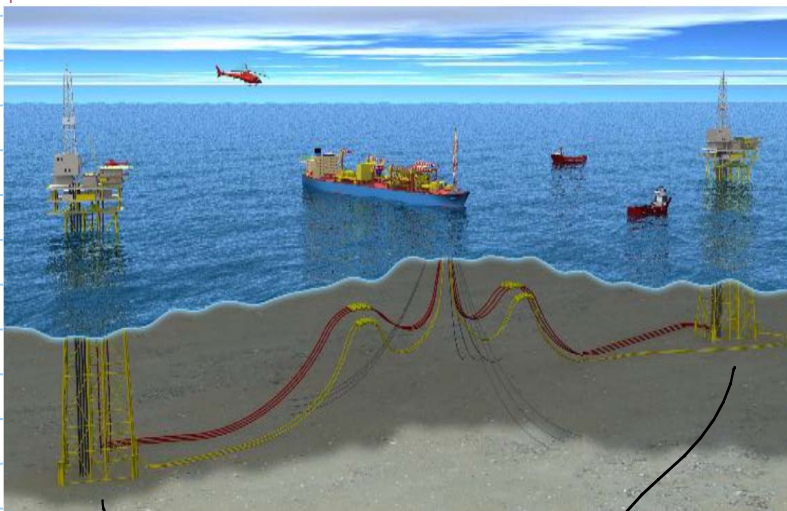
API : 14

$\mu_o = 120 - 200 \text{ cp}$

@ Tr

$GOR \approx 100 \text{ Sm}^3/\text{Sm}^3$

agiter and water reinjection

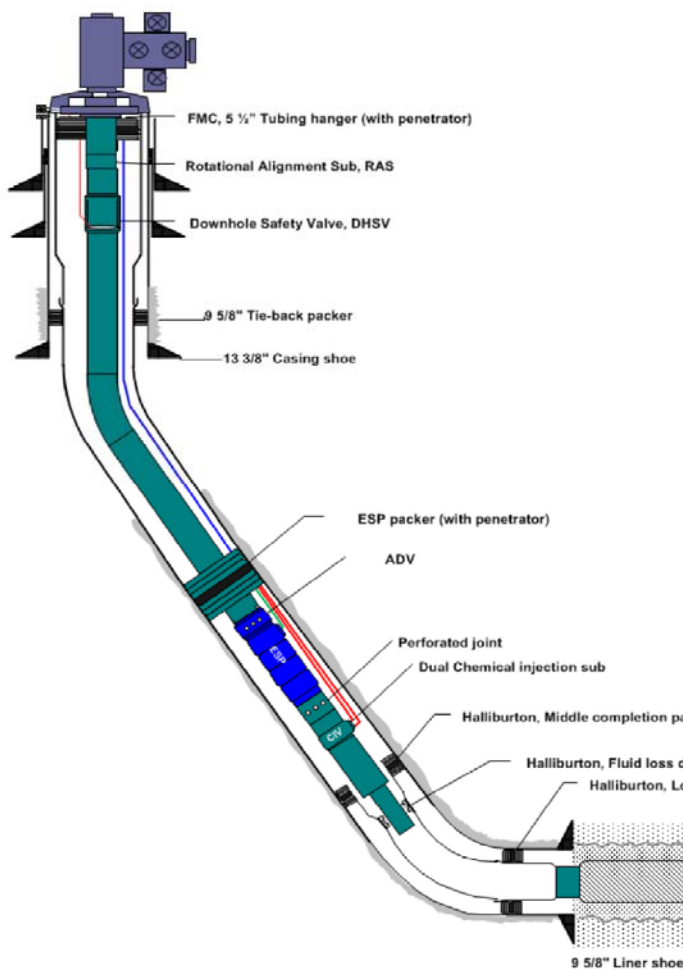
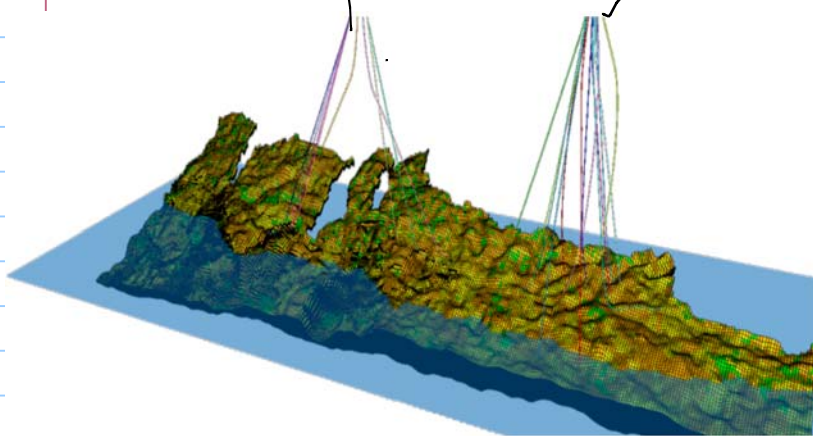


- LQ
- Drilling module
- Movable drill rig
- Electrical plant
- Wellheads
- Booster pumps
- Water Injection
- Uptime 99% bracket

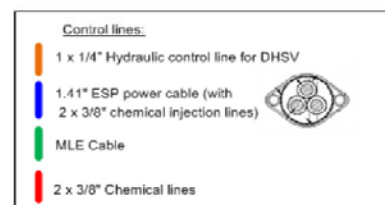
typical life of ESP (5 years)
medium life (2 years - 1 year)

• frequent startups ~ shutdowns hurt the life of the pump.

• Sand production



Section	Casing/Liner Size	Weight (lb/ft)	Grade	Threads
26"	26"	267.2#	X-56	D-90MT
22"	18 1/2"	87.5#	J-55	TenarisER
17 1/2"	13 1/2"	68#	L80	VamTop
12 1/4"	10 3/4"	45.5#	L80	VamTop
	9 1/2"	47#	L80 & 13Cr80	VamTop
8 1/2" Branch Y1	5 1/2" Screens & blank pipe	17#	13Cr80	VamTop
Mid. Completion	5 1/2" blank pipe	17#	13Cr80	VamTop
Tubing	5 1/2"	17#	L80	VamTop

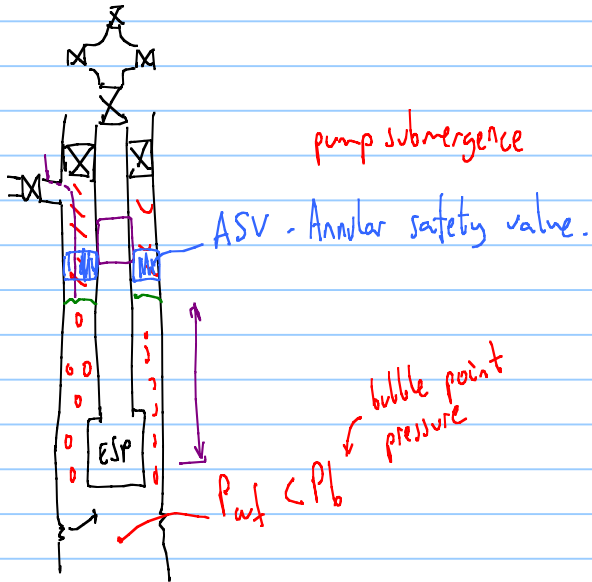


Halliburton PetroGuard 5 1/2" wire wrapped screens
5 1/2", 17#, 13%Cr basepipe. 316L Wrap w/ AICD
Gravel packed 8 1/2" hole

ESP lifted wells

liquid through tubing
gas through annulus

liquid and gas (if any) through tubing



gas affects negatively the performance of the pump

- ↳ performance
- ↳ vibration
- ↳ cooling
- ↳ frequent failure

$$P_{suc} \geq F_s \cdot P_b$$

$$P_{suc} \geq 1.05 P_b$$

7/ 1.25 30 bar

GVF $\rightarrow$ gas volume fraction

$$GVF = \frac{q_g \overset{\text{local}}{\leq 0.3}}{q_L + q_g}$$

$$q_c = J_c \cdot (P_n - P_{wf})$$

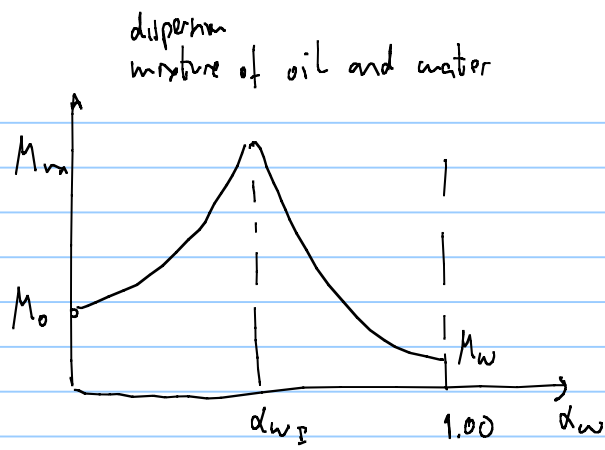
Wells	P_R	J, for total liquid flow	h_1	h_2	d_1	d_2	Pumpname	Stages
-	bara	$\text{Sm}^3/\text{d}/\text{bar}$	m	m	m	m	-	-
1	231	14	380	1960	0.24	0.14		78
	Water	Oil						
Fluid Density	1025	897 [kg/m ³]						
Richardson Emul. exp.	3.089	3.215						
Viscosity	1.00E-03	0.1 [Pa s]						
α_w cut off	0.60							
Roughness tubing and lines	0.00010 m							
Bubble point pressure [bara]	30 [bara]							

h_1 .. distance from sand phase to pump suction
 h_2 .. distance from pump discharge to well head

h_1 -- distance from sand phase to pump section

h_2 .. distance from pump discharge to well head

[illegible]

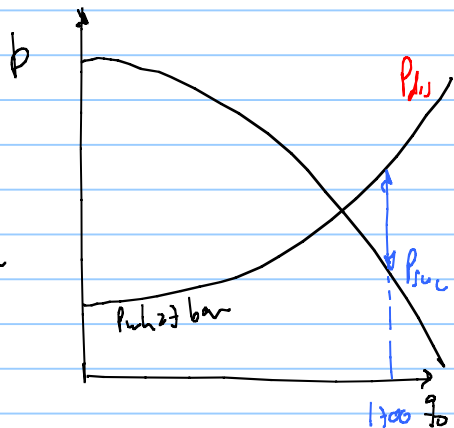


$$q_{\text{target}} = 1700 \text{ m}^3/\text{d oil}$$



$$\Delta p_{\text{pm}} (P_{o,i} - P_{suc}) @ q_o = 1700 \text{ m}^3/\text{d}$$

$$R_L \quad q_o$$



$$\mu_m = \mu_o (1 - \alpha_w) + \mu_w (\alpha_w)$$

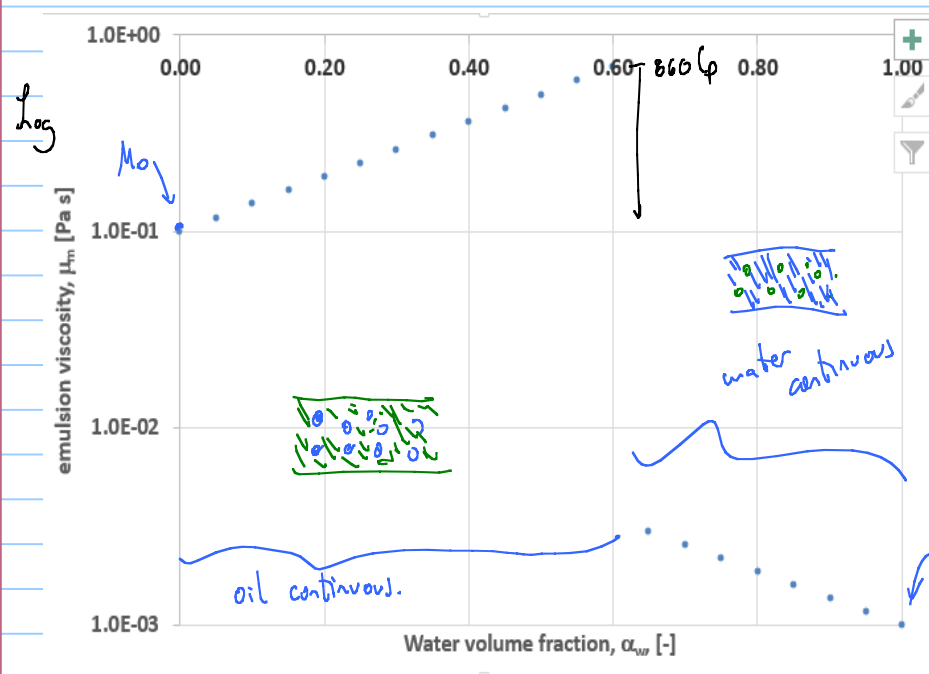
assumption

the oil and water are incompressible

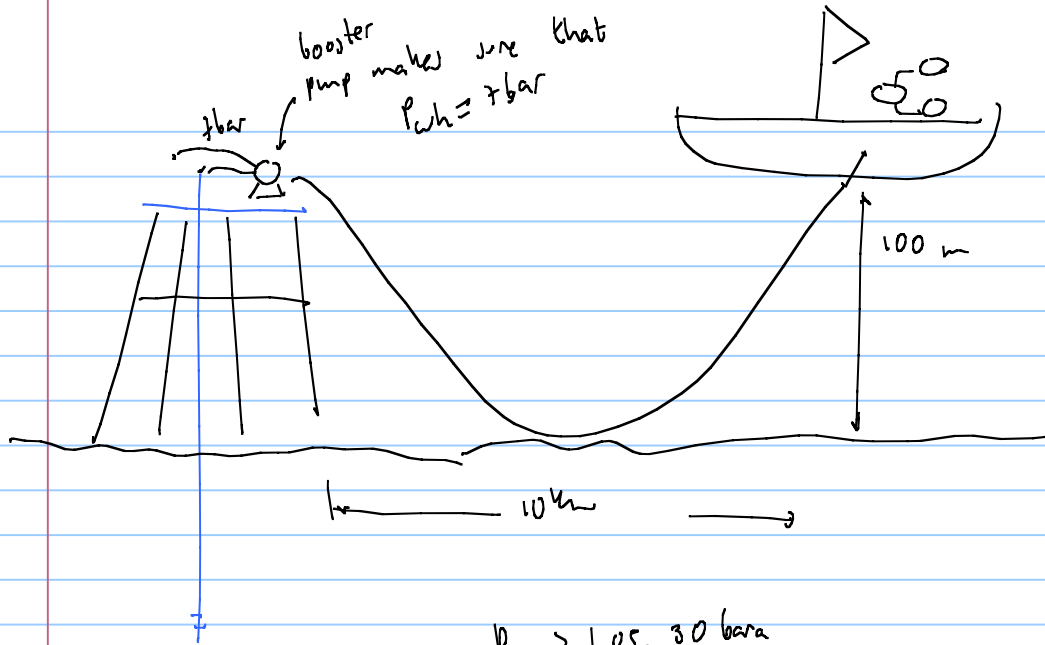
$$q_o = q_o(P, T)$$

$$\alpha_w(P, T) = wC$$

$$q_w = q_w(P, T)$$



emulsion behavior.



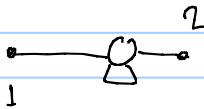
$$P_{suc} \geq 1.05 \cdot 30 \text{ bara}$$

$$P_{suc} \geq 31.5 \text{ bara}$$

Power capacity $\rightarrow$ casing size limits the maximum power capacity
 $\searrow$ $\uparrow$

300 hp $\rightsquigarrow$ 1500 hp
 minimum maximum

600 hp $\rightarrow$ comes from the experts



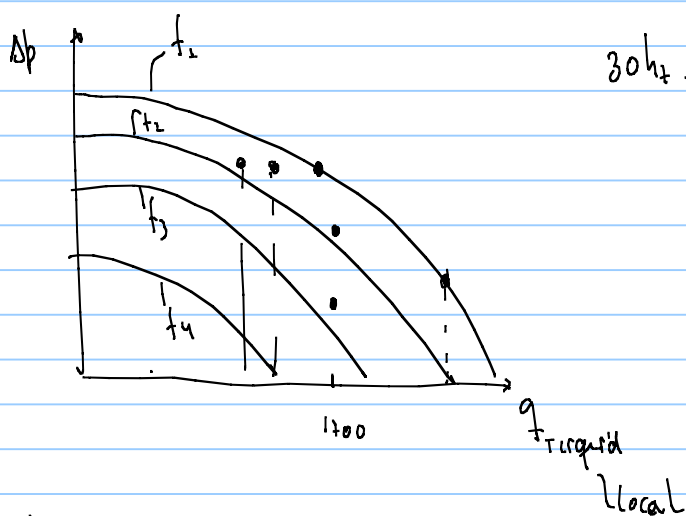
$$P_{Hydraulic} = \frac{(P_2 - P_1) \cdot Q}{\eta_{eff}} \uparrow$$

$\eta_{eff} \rightsquigarrow ?$ $50\% \sim 70\%$
 $\rightarrow$ assume 60%

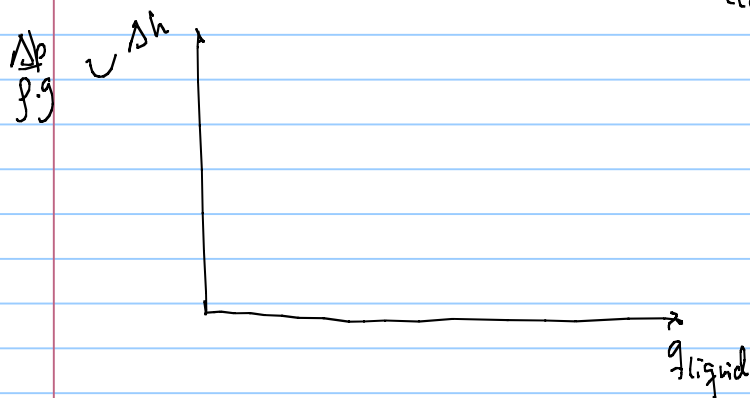
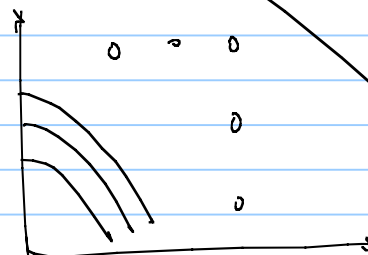
$$\dot{m} = \frac{Q}{s} = \frac{N \cdot m}{s} =$$

$$P_a = \frac{m^3}{s} = \frac{N}{m^2} \frac{m^3}{s} = \frac{J}{s}$$

-	bara	Sm ³ /d/bar	m	m	m	m	-	-			
1	231	14	380	1960	0.24	0.14	-	78			
	Water	Oil									
Fluid Density	1025	897	[kg/m ³]								
Richardson Emul. exp.	3.089	3.215									
Viscosity	1.00E-03	0.1	[Pa s]								
α_w cut off	0.60										
Roughness tubing and lines	0.00010	m									
Bubble point pressure [bara]	30	[bara]									
Wells	P_R	WC	Average density	Effective viscosity	q_{tot}	P_{wf}	q_o	q_w	P_{suc}	P_{disc}	ΔP_{ESP}
-	bara	[-]	kg/m ³	Pa s	Sm ³ /d	bara	Sm ³ /d	Sm ³ /d	bara	bara	bara
	231	0.00	897	0.100	1700	110	1700	0	76	185	10
	215	0.00	897	0.100	1700	94	1700	0	60	185	12
	200	0.00	897	0.100	1676	80	1676	0	47	185	13
	175	0.00	897	0.100	1527	66	1527	0	32	184	15
	150	0.00	897	0.100	1190	65	1190	0	31.5	183	15

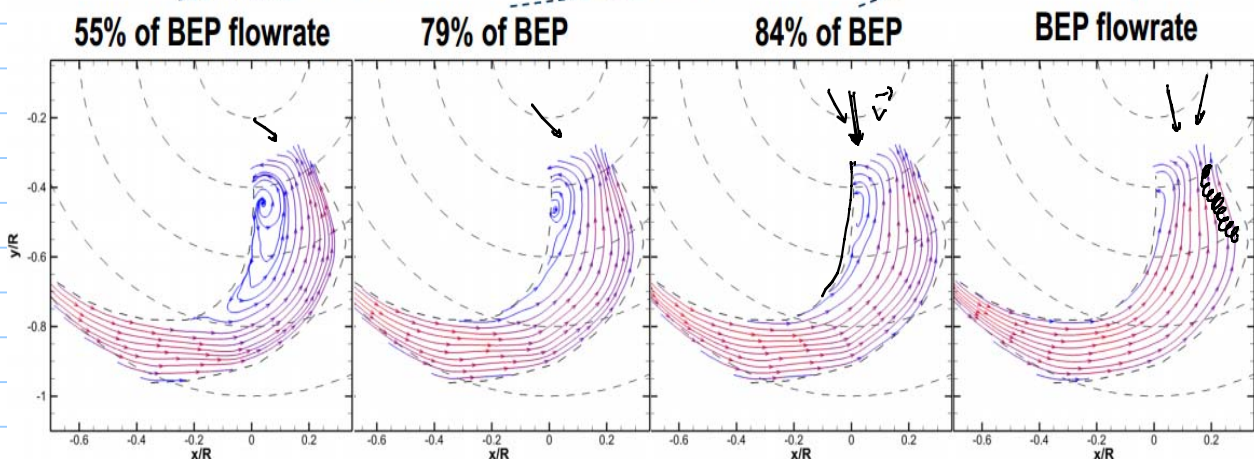
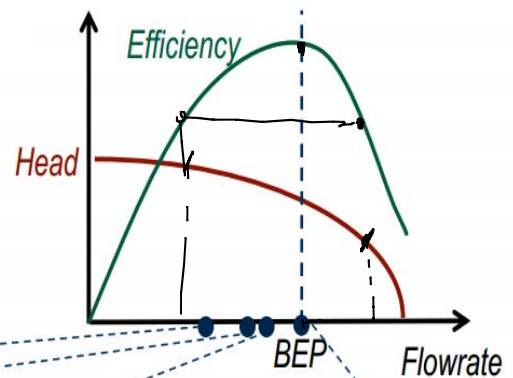


$$30 h_1 \leq f \leq 60 h_2$$



PIV measurement in a radial flow stage

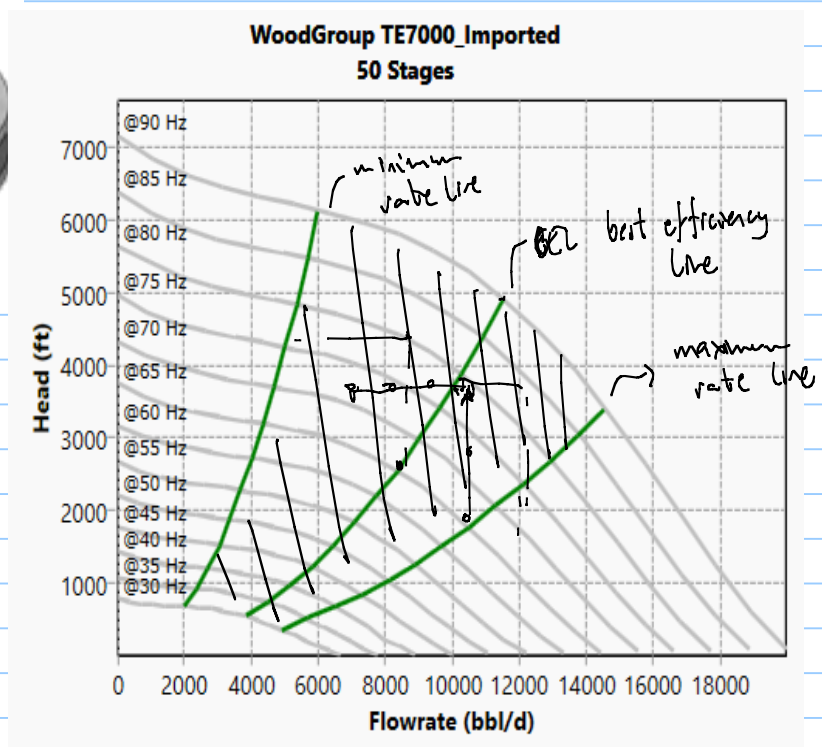
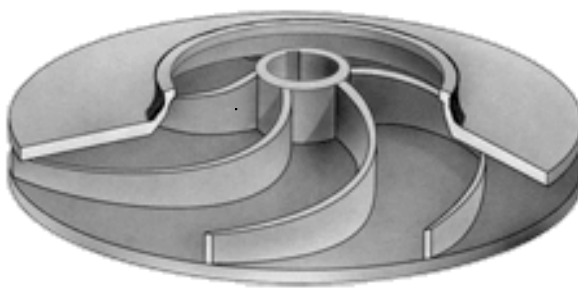
- Flow features in diffuser and impeller may be identified from measurements
- Flow misalignment and recirculations reduce efficiency

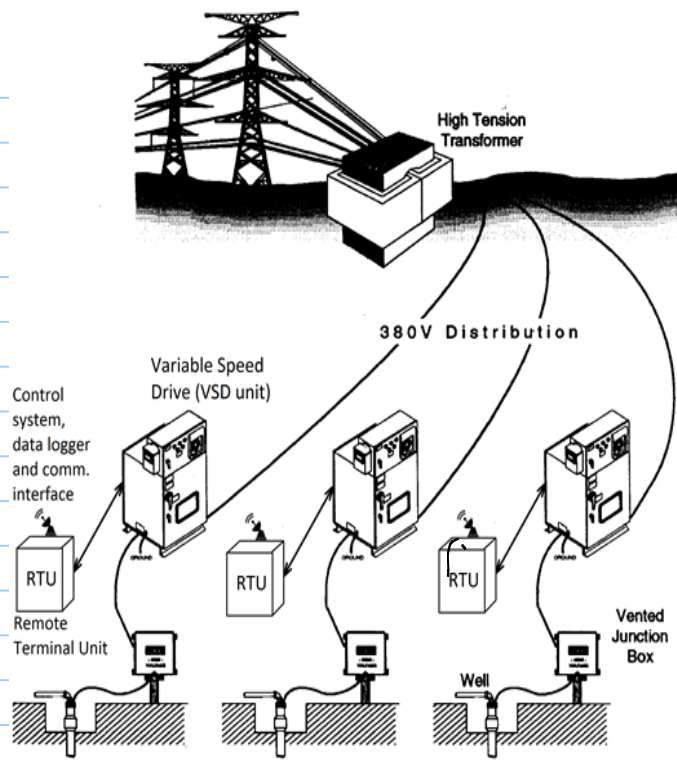


Example of stall region in diffuser passage (measured)

SPE-14MEAL-14017-PP-MS • Measurement and Unsteady Simulation of Internal Flows within Stages • J Dusting

BEP when the flow is aligned with the vane angle





$$1700 \frac{m^3}{d}$$